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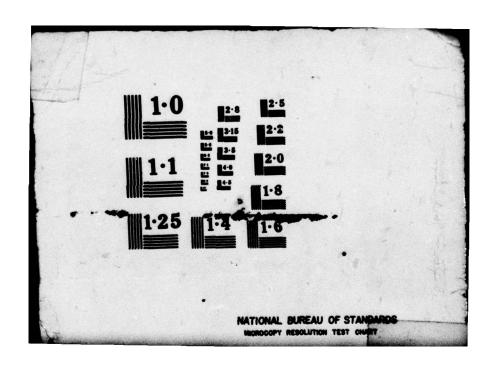
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Crew Equipment and Recovery Branch Strategic Systems Special Projects Office

May 1979

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10 Dennis W. /Schroll

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10. ABSTRACT (Continue on reverse side if necessary and identify by block number)

Much research and testing has been accomplished in the past years to develope a supersonic ramjet engine. The topic is discussed in almost all propulsion texts. However, very little headway has been made in theoretical design techniques as many of the standard computational methods used for ramjet subsonic combustion do not apply. For instance, it is possible to use a constant area nozzle for subsonic combustion in a ramjet as heat

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addition and frictional effects in the combustion chamber will decrease the pressure and accelerate the flow. For supersonic combustion the velocity decreases tending to choke the flow. . To overcome this difficulty area increase in the combustion zone diffuser inlet for free stream mach numbers of 5 and 10 and the analysis also considers shock free flow of these mach numbers and assumed Accompanied with the strong normal shock are large stagnation pressure losses in the diffuser inlet, meaning the diffuser inlet will act as a flat plate to oncoming airflow. For this reason, it would be best to provide air spillage and not use too large an inlet. For the shock expelled case, the drag through the engine would be less, but static pressure rises up to 272 atmospheres are calculated.

REPORT DOCUMENTATION FIRST

Basic assumptions that apply to the computational schemes in this paper are the quantum mechanical relationships developed by statistical mechanics. This means the assumption of equilibrium must apply within the zones of the engine calculated.

This is a most valid assumption for high speed flow. High energy molecules will have time to reach equilibrium values along each section of the engine, but have little time to dissipate to the walls of the engine. The inviscid assumption is then valid. Of course, chemical reactions within the combustion chamber are not so simple in nature as applied here, but the constant pressure assumption allows chemically reacting computations to be determined and then compared to the perfect gas computational

The final Thrust Specific Fuel Comsumption (TSFC) values are sufficiently high to warrant further investigation into supersonic combustion as a method of propulsion. They are slightly higher than that of a designed ramjet which is in operation with hydrocarbon fuels. At these very high speeds, it is possible to pass a larger amount of air mass per unit time through the supersonic ramjet engine. It's thrust values will therefore be much higher than for the developed ramjets.

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#### FOREWORD

This report is the compiled results of an in-house research project conducted by the author, Dennis W. Schroll, while attending the Ohio State University, Columbus, Ohio. This final research report was to fulfill the requirements for a Master of Science Degree in Aeronautics and Astronautics under the sponsorship of Aerospace Systems Division (DPCD) Long-Term Full-Time Training Contract F3360-75-A-0549-0002.

The work reported herein was performed during the period March 1978 to October 1978 by the author. This report was released by the author in December 1978.

The author wishes to give special thanks to his immediate supervisor at the time, Mr. W. A. Lucka, for his initial nomination to this program and to Professor R. Edse for being research advisor. This study would not have been possible without Professor Edse's equations and tables which he has developed in his research projects that are closely related to those outlined in this report.

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# GLOSSARY OF TERMINOLOGY

	4일 이 10 10 10 10 10 10 10 10 10 10 10 10 10
T	(°K) temperature
P	(atm) pressure
u	(m/sec) velocity
M	dimensionless frozen flow mach number
ni	partial pressure fractions of molecules and atoms
m	(g/mole) molecular weight
SR2	dimensionless entropy
hf abs T	dimensionless enthalpy
A	(m <sup>2</sup> ) area
8	ratio of specific heats for frozen flow
n(i)	global molar value of species i
æ	universal gas constant = 8314.33 J/K-mole OK
K <sub>p</sub> (i)	equilibrium pressure coefficient
Hfli	absolute enthalpy of each species
$\left[\frac{C_{D}}{R}\right]_{i}$	non-dimensionalized specific heat
hf abs	absolute formation enthalpy
To, Po	stagnation fluid properties determined from the isentropic perfect gas relationships
9	density
9	heat addition coefficient
J 02°	fuel-air ratio
x 2	correction coefficient

## GLOSSARY OF TERMINOLOGY (continued)

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delines to sulay raion laidle

x<sub>i</sub> mass fraction of species i

precific net thrust

ma

TSFC thrust specific fuel consumption

f mass weighted fuel-air ratio

significan fluid propertiesed acceptable from the isentropic

#### SECTION I

#### INTRODUCTION

A variable area supersonic combustion ramjet was modeled for free stream mach numbers of 5 and 10. It was desired to use a configuration as that proposed in several articles on a hypersonic research vehicle as a joint NASA-USAF project. The problem is divided into four sections: 1) the starting shock at M<sub>2</sub> =5, 2) the swallowed shock at M<sub>2</sub> =5, 3) the starting shock at M<sub>3</sub> =10, 4) the swallowed shock at M<sub>4</sub> =10.

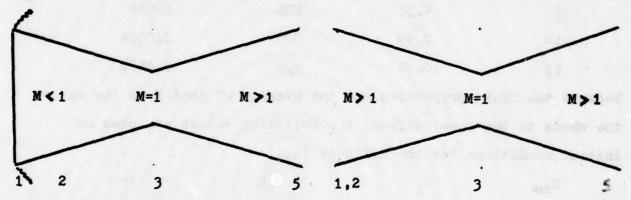


Figure 1. Entrance Shock for Starting Ramjet-Subcritical Condition

Figure 2. Shock is Swallowed with Supersonic Flow Throughout-Supercritical Condition

The flow is considered non-viscous and the geometry of the converging-diverging channels are 2 dimensional and planar. Effects of curvature to compress and expand the flow uniformly across incremental mach lines are ignored. Rather, it is the purpose of this project to determine the fluid properties at various stations through constant properties of entropy and pressure. Dissociation of air and hydrogen/air chemical reactions are computed.

### SECTION II

## COMPRESSION THROUGH VEHICLE BOW SHOCK WAVE

The bow shock wave off the hypersonic vehicle is considered as the first stage of air compression. From Taylor-Maccoll cone theory (cone at zero angle-of-attack) with a  $7\frac{1}{2}$  degree cone half angle, fluid properties are estimated to be:

No to	owelle M1 ads (A	T <sub>1</sub> OK	P <sub>1</sub> atm
5	4.51	236	0.018
10	7.99	306	0.0374
15	10.37	408	0.0687

Because the fluid properties are the average of that from the ray on the shock to the cone surface, the following values are used as initial conditions for the diffuser inlet:

Moo	M <sub>1</sub>	T <sub>1</sub> °K	P <sub>1</sub> atm
Solling at a	5	230	0.02
10	8	300	0.04

Thus the bow shock wave off the vehicle acts as a mild first stage compressor.

AIR IS 
$$\longrightarrow$$
 0<sub>2</sub> + 3.76 N<sub>2</sub>  
 $=$  8314.33 Joules/K-mole <sup>o</sup>K

The molar volume of air of 3.76 of nitrogen to 1 of oxygen is assumed and the value of the universal gas constant as used through this report is given.

## SECTION III

## PERFECT GAS COMPUTATIONS ACROSS DIFFUSER INLET SPIKE

To estimate the fluid properties temperature and pressure across the shock, perfect gas relationships are used with decreasing values of X for higher temperatures.

$$P_{2} = P_{1} \left[ \frac{2 \times 1}{3 + 1} M_{1}^{2} - \frac{3 - 1}{3 + 1} \right]$$

$$T_{2} = T_{1} \left[ \frac{(1 + \frac{3 - 1}{2} M_{1}^{2})(3 M_{1}^{2} - \frac{3 - 1}{2})}{(\frac{3 + 1}{2} M_{1})^{2}} \right]$$
Shock wave

RAY

Cone

Page

Page

Figure 3. Inlet Spike or Forward Region of Hypersonic Vehicle

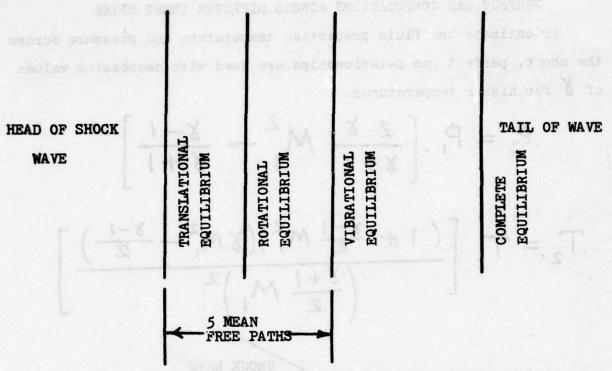


Figure 4. Internal Modes of Energy of Diatomic Oxygen and Nitrogen Showing Regions of Relaxion

Values of X for estimated temperatures can be determined approximately by guessing that the values begin at 1.4 for air  $(0_2 + 3.76 \text{ N}_2)$  and decrease with increasing temperatures where dissociation begins at 1100 °K. When  $M_1=8$  for  $T_1=300$  °K and X=1.3,  $T_2=3300$  °K, however this does not consider molecular dissociation and will need to be adjusted.

## SECTION IV

## AIR DISSOCIATION THROUGH A SHOCK WAVE

To adjust the temperature and pressure it is necessary to calculate the mole fractions of molecule and atom species of  $0_2$ ,  $N_2$ , 0, N and NO from the estimated temperature and pressure. The equilibrium pressure

coefficient equations of each individual reaction is written,  $(K^{(i)})$  and the fact that the sum of the mole fractions is equal to 1, and the ratio of oxygen to nitrogen atoms is given. The equations are rearranged and values of  $\chi_{0_2}$  are estimated until the percent error is less than .001.

$$\eta_{o} = K^{(0)} \sqrt{\Omega_{o_{2}}}$$

$$\sqrt{N_{N_{2}}} = \sqrt{\alpha^{2} + 1 - N_{o_{2}} - N_{o}} - \alpha$$
where
$$\alpha = \frac{K^{(N)} + K^{(NO)} \sqrt{N_{o_{2}}}}{2}$$
values of  $N_{o_{2}}$  are assumed until

$$\frac{100}{3.76} \left| \frac{N_{N_{2}} + \frac{1}{2}(N_{N} + N_{NO})}{N_{o_{2}} + \frac{1}{2}(N_{o} + N_{NO})} - 3.76 \right| = |\Delta| < .001$$
and
$$\eta_{NO} = \sqrt{N_{N_{2}}} N_{o_{2}} K_{P} (NO)$$

$$\eta_{N} = \sqrt{N_{N_{2}}} K_{P} (NO)$$

The Hugeniot Equation is solved for P2 EST until the per cent error is less than .01%.

$$\frac{P_{1}}{P_{1}} = B + \sqrt{B^{2} + \frac{T_{2}}{T_{1}} \cdot \frac{m_{1}}{m_{2}}}$$
where
$$B = \frac{T_{2}}{T_{1}} \cdot \frac{m_{1}}{m_{2}} \sum_{i} n_{i,2} \left(\frac{H_{f}}{RT}\right)_{i}^{T_{2}} - \sum_{i} n_{i,1} \left(\frac{H_{f}}{RT}\right)_{i}^{T_{1}} \cdot \frac{\left(\frac{T_{2}}{T_{1}} \cdot \frac{m_{1}}{m_{2}} - 1\right)}{\left(\frac{T_{1}}{T_{1}} \cdot \frac{m_{2}}{m_{2}} - 1\right)}$$

It is now necessary to recalculated the molar fractions each time a new pressure is assumed. Now with these series of equations solved for an estimated temperature and corresponding values of pressure, a value of the velocity is calculated.

$$u_{2} = \frac{T_{2}}{T_{1}} \cdot \frac{m_{1}}{m_{2}} \cdot \frac{P_{1}}{P_{2}} U_{1}$$

$$\left[ U_{1}^{\text{calc}} \right]^{2} = \left[ \left( \frac{P_{2}}{P_{1}} - I \right) \div \left( I - \frac{T_{2}}{T_{1}} \cdot \frac{P_{2}}{P_{1}} \right) \right] \left[ \frac{T_{1}^{2} m_{2} \mathcal{R}}{m_{1}} \right]$$

$$until \left| \frac{U_{1}^{\text{calc}} - U_{1}^{\text{calc}} + U_{1}^{\text{calc}}}{100 \times U_{1}^{\text{calc}} + U_{1}^{\text{calc}}} \right| < .01\%$$

By successive linear extrapolation a new value of temperature is determined, the pressure and mole fractions are adjusted until the correct velocity is calculated. This gives values of  $T_2$ ,  $P_2$  and  $u_2$  behind the shock.

Now consider frozen flow where a value of is determined for these values of temperature and pressure.

$$y^{\text{FMOZ}} = \frac{\sum_{i} n_{i} \frac{c_{i}}{n_{i}}^{T}}{\sum_{i} n_{i} \frac{c_{i}}{n_{i}}^{T}} - 1$$

To calculate the mach number (MFROZ), it is necessary to determine the speed of sound in the mixture at the position behind the shock. For temperatures less than 3000 °K, the contributions of the shifting speed of sound are neglible and only the frozen speed of sound is considered. As such the reference mach number is calculated from the definition of the speed of sound.

at M<sub>1</sub>=8, the value of the frozen mach number behind the inlet shock is

$$M^{FROZ} = \frac{2785}{\sqrt{1.2891 \left(\frac{8314.35}{28.36149}\right) 3.089}} = 0.3413$$

Now that the fluid properties are determined across the non-isentropic strong normal shock, a question might be posed as to whether the condition of assumed equilibrium is valid. Those modes of energy such as translational, rotational for diatomic oxygen and nitrogen usually reach equilibrium in less than 5 mean free paths of collision of the molecules. Relaxation of the vibrational energy state usually takes a somewhat longer time, but for oxygen and nitrogen relaxation still occurs in a narrow region behind the shock. Therefore the frozen properties are assumed to be determined after relaxation or for equilibrium ( see Figure 4 ).

#### SECTION V

#### ISENTROPIC DIFFUSER FLOW CALCULTIONS

Behind the shock wave the flow is subsonic and a converging channel will increase the flow velocity, and decrease the molecule translational energy (temperature) and decrease the pressure. That is, the more random molecular motion will be more ordered and directional. The temperature and pressure are estimated at the throat to be values as determined from the isentropic perfect gas relationships

$$\frac{\frac{R}{R_3}}{P_3} = \left(1 + \frac{y-1}{2} M_3^2\right)^{\frac{y}{y-1}} \frac{T_0}{T_3} = \left(1 + \frac{y-1}{2} M_3^2\right)$$

It is not necessary to calculate the stagnation properties as ratios can be taken and with sonic velocity assumed at the throat the equations are

equations are
$$P_{3} = P_{2} \left\{ \frac{(1 + \frac{Y-I}{2}M_{2}^{2})}{\frac{Y+I}{2}} \right\}^{\frac{Y-I}{2}} T_{3} = T_{2} \left\{ \frac{1 + \frac{Y-I}{2}M_{2}^{2}}{\frac{Y+I}{2}} \right\}$$

It is now necessary to adjust the pressure and temperature to account for the dissociation of the air. The equation of entropy is

$$\frac{s}{R_2}\Big|_{zsr}^{T_{esr}} = \frac{m_2}{m} \Big\{ \sum_{i} \eta_i \Big( \frac{s}{R} \Big)^{T_{esr}} - \sum_{i} \eta_i \ln \eta_i - \ln P \Big\}$$

with the known properties behind the shock the dimensionless entropy is calculated. Next calculate the entropy for the estimated isentropic temperature and pressure. To do this the mole fractions are calculated as previously outlined and then the entropy is adjusted by changing the pressure until it is within .01% that of the entropy behind the shock. Each time a new pressure is assumed new values of

mole fractions must be calculated. However the mole fractions are relatively insensitive to small changes in pressure. To adjust the temperature, the absolute enthalpy is calculated.

$$\left(\frac{h_{f abs}}{R_{2} T_{2}}\right)^{T_{EST}} = \frac{T_{EST}}{T_{2}} \frac{m}{m} \sum_{i} \eta_{i} \left(\frac{\Delta H_{f abs}}{R}\right)^{T_{EST}}$$

From the energy equation for no fuel addition to the flow.

$$h_{fabs} = h_{fabs} + \frac{u_{z}^{2}}{2} = h_{fabs}^{T_{3}} + \frac{u_{s}^{2}}{2}$$

$$u_{s}^{calc} = \left\{ \left[ \left( \frac{h_{fabs}}{R_{z} T_{z}} \right)^{T_{z}} - \left( \frac{h_{fabs}}{R_{z} T_{z}} \right)^{T_{s}} \right] \frac{L(8314.35) T_{z}}{2} + u_{z}^{2} \right\}$$

$$v_{s}^{fRoz} = \frac{\sum_{i} n_{i} \frac{c_{f}}{R_{z}} v_{s}^{T_{s}}}{\sum_{i} n_{i} \frac{c_{f}}{R_{z}} v_{s}^{T_{s}}} = \frac{\sum_{i} n_{i} \frac{c_{f}}{R_{z}} v_{s}^{T_{s}}}{\sum_{i} n_{i} \frac{c_{f}}{R_{z}} v_{s}^{T_{s}}} = \frac{u_{s}^{calc}}{\sum_{i} n_{i} \frac{c_{f}}{R_{z}$$

 $M_3^{FROZ} = \frac{U_3^{CALC}}{\sqrt{\chi_2^{FROZ} \frac{8314.35}{m_2} T_3^2}}$ 

Should the mach number be larger than 1, then choose a value of temperature 100 °K larger and repeat the above procedure until a new mach number is determined. Now by linear extrapolation a third more accurate temperature is determined.

$$T_3^r (est 3) = T_3^{est 1} + [M_3 = 1 - M_3^{est 1}]$$

$$\times \left[ \frac{T_3^{est 2} - T_3^{est 1}}{M_3^{est 2} - M_3^{est 2}} \right]$$

The calculations are now repeated again and usually this value is within 0.1% error as the frozen flow mach number relationship is a fairly linear one with respect to temperature for constant entropy.

## SECTION VI

## ISENTROPIC DIFFUSER EXIT FLOW CALCULATIONS

To go to region 3C (exit diffuser and entrance of combustion chamber) is somewhat more involved. Heat addition due to the combustion of hydrogen and air for various values of the heat addition coefficient are calculated.

Now 
$$\frac{T_{3c}}{T_{5}} = \frac{1 + \frac{Y-1}{2} M_{3}^{2}}{1 + \frac{Y-1}{2} M_{3c}^{2}}$$
for isentropic expansion, from

3 to 3C and for dP=0, du=0 from 3C to 4.

$$M_{4} = \frac{M_{3C}}{\sqrt{1 + \frac{9}{c_{p}}T_{3C}}} \frac{T_{04}}{T_{03C}} = 1 + \frac{\frac{9}{c_{p}}T_{3C}}{1 + \frac{8-1}{2}M_{3C}}$$

$$\frac{P_{04}}{P_{05C}} = \left[\frac{1 + \frac{9-1}{2}M_{3C}}{1 + \frac{9-1}{2}M_{3C}}\right]^{\frac{8}{8}} - 1$$

$$\frac{A_{4}}{A_{3C}} = \frac{T_{4}}{T_{3C}} = 1 + \frac{9}{c_{p}}T_{5C}}{T_{5C}} = 1$$

The heat addition coefficient is then non-dimensionalized with respect to T<sub>3</sub> and plots are made.

$$M_{4} = \frac{M_{3C}}{\sqrt{1 + \frac{q}{C_{\rho}T_{3}}} \left[ \frac{1 + \frac{y-1}{2} M_{3C}^{2}}{\frac{1+y}{2}} \right]} \frac{1}{\sqrt{1 + \frac{q}{C_{\rho}T_{3}}} \left[ \frac{1 + \frac{y-1}{2} M_{3C}^{2}}{\frac{1+y}{2}} \right]} \frac{y}{\sqrt{1 + \frac{q}{C_{\rho}T_{3}}}} \frac{y}{\sqrt{1 + \frac{y-1}{2} M_{3C}^{2}}} \frac{y}{\sqrt{1 + \frac{y-1}{2} M_{3C}^{2}}}} \frac{y}{\sqrt{1 + \frac{y-1}{2} M_{3C}^{2}}}} \frac{y}{\sqrt{1 + \frac{y-1}{2} M_{3C}^{2}}} \frac{y}{\sqrt{1 + \frac{y-1}{2} M_{3C}^{2}}}} \frac{y}{\sqrt{1 + \frac{y-1}{2} M_{3C}^{2}}} \frac{y}{\sqrt{1 + \frac{y-1}{2} M_{3C}^{2}}}} \frac{y}{\sqrt{1 + \frac{y-1}{2} M_$$

The non-dimensionalized heat addition coefficient q is given above. See reference 1 for an outline of the procedure to obtain this coefficient.

For hydrogen

The mass of hydrogen-air mixture can be expressed relative to the number of moles of oxygen by  $\sqrt{0_2}^\circ$ . Before combustion this ratio is given by

$$H_2 + \sqrt{0_2}^{\circ} (0_2 + 3.76 N_2)$$

The maximum heat release is obtained for  $v_{0_2}^{\circ} = 0.5$  as the oxygen-hydrogen chemical reaction is  $H_2 + \frac{1}{2}O_2 = H_2O$ .

However, it is possible to have a leaner fuel-air mixture ratio  $\frac{1}{2} \leq \sqrt{0}$   $0 \leq \infty$  .

The specific heat of the mixture, m is given as

$$C_{P,m} = \frac{\chi_{m}}{\chi_{m-1}} R_{m}$$
 where  $R_{m} = R/M_{m}$  and =8314.33 is the Universal Gas Constant

Several ideal assumptions are made in the combustion chamber that expedite computations. The hydrogen enters the combustion chamber at the same speed as that of the air, and its absolute enthalopy is determined at 1000°K for temperatures exceeding this and at the air temperature for air temperatures less than this

Now according to the method outlined in reference 1 the value of

$$m_{m} = \frac{4.76 \, v_{o_{2}} \, \frac{m_{H_{2}}}{m_{o_{2}} + 3.76 \, m_{H_{2}}}}{1 + 4.76 \, v_{o_{2}} \, \frac{m_{o_{2}} + 3.76 \, m_{H_{2}}}{m_{1}}} (m_{o_{2}} + 3.76 \, m_{N_{2}})$$

For temperatures less than 2000  $^{\circ}$ K the mass of the decelerated air is almost equal to  $m_{0_2}$  + 3.76 $m_{N_2}$ . Substituting in all the approprite values and cancelling out gives,

 $\frac{q}{C_{for}T_{i}} = \chi \frac{y-1}{8} \frac{29048}{(4.76)_{o_{2}}^{\circ} + 1) T_{3}^{\circ}}$ 

for hydrogen air mixture will range from 1.441 (no combustion) to 1.25 (combustion temperatures up to 3000 °K). Since this non-dimensionalized heat coefficient is mainly used to get an estimate of M<sub>3C</sub>, = 1.4 is used as this is at the position of combustion initiation. A few words should be said about the correction coefficient X, as this pertains to the degree of combustion of gases as they travel across the nozzle. It can range from 0.4 for high temperatures and low pressures up to 1.0 for low temperatures and high pressures.

With the possible exception of one case M<sub>20</sub> = 10, no shock, values of 0.7 would seem satisfactory. For the sake of comparison, 0.7 is used in all computations. Two combustion cases were taken for all computations.

 $\int_{0_2}^{0} = 0.5$  (rich mixture - maximum heat release)  $\int_{0_2}^{0} = 2$  (lean mixture - minimum heat release)

The procedure to determine the fluid properties at the diffuser exit

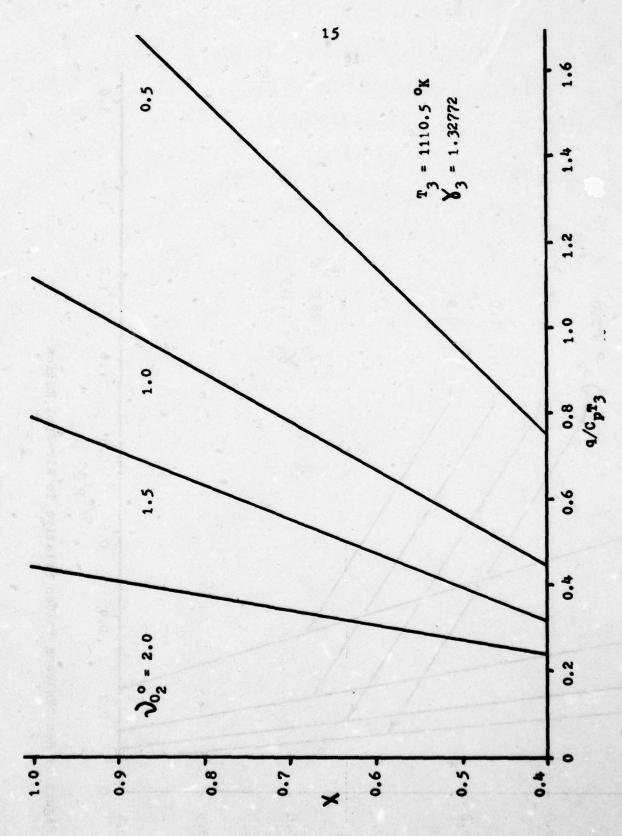
3C was to first of all assume we want  $M_{i,j} = 1.2$ . Too high a mach number in the combustion chamber will mean problems with mixing fuel and air. However, should the flow go subsonic again this would mean flow choking (M = 1) and the flow in the combustion chamber will go to low mach numbers. Therefore, in conclusion, it is desired to keep the flow within the mach number range at  $M_3 = 1$  at the throat to  $M_{i,j} = 1.2$  at the exit of the combustion chamber.

Looking at Figure 3.  $M_3$  can be determined from a calculated heat release coefficient and assumed value of  $M_4$  = 1.2. This mach number is then used to calculate the fluid properties isentropically at the diffuser exit.

$$T_{3c} = \frac{T_{3c}}{T_{o3c}} \frac{T_{o3}}{T_{7}} T_{7} = \frac{\left[1 + \frac{y-1}{2} M_{3}^{2}\right] T_{7}}{\left[1 + \frac{y-1}{2} M_{3c}^{2}\right]} T_{7}$$

$$P_{3c} = \frac{P_{3c}}{P_{o3c}} \frac{P_{o3}}{P_{7}} P_{7} = \frac{\left[1 + \frac{y-1}{2} M_{3c}^{2}\right] Y_{7}}{\left[1 + \frac{y-1}{2} M_{3c}^{2}\right]} Y_{7} - 1 P_{7}$$

Next, calculate the flow properties considering air dissociation by the same method as used to determine the flow properties at the throat until the value of M<sub>3C</sub> FROZ is within 0.1% error of that of the mach number determined by the perfect gas method. Inspection of Figures 5 through 10 reveals that it is necessary to add fuel and combust the gas downstream of the throat to prevent choking the flow. Even though the molecular velocity will approximately double in magnitude from the diffuser throat to the diffuser exit for supersonic flow, the expanding nozzle will result in a decrease in temperature and pressure



Pigure 5. Heat Release Values Relative to Air-Fuel Ratios

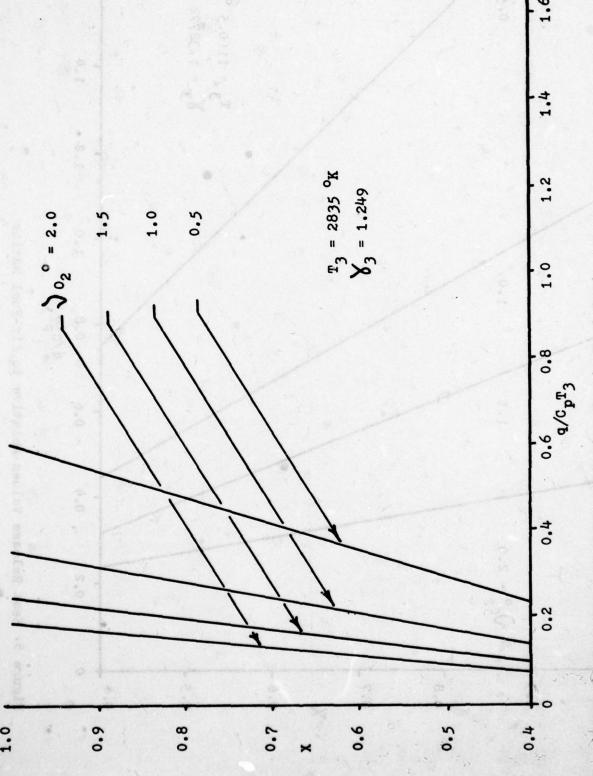


Figure 6. Heat Release Values Relative To Air-Fuel Ratios

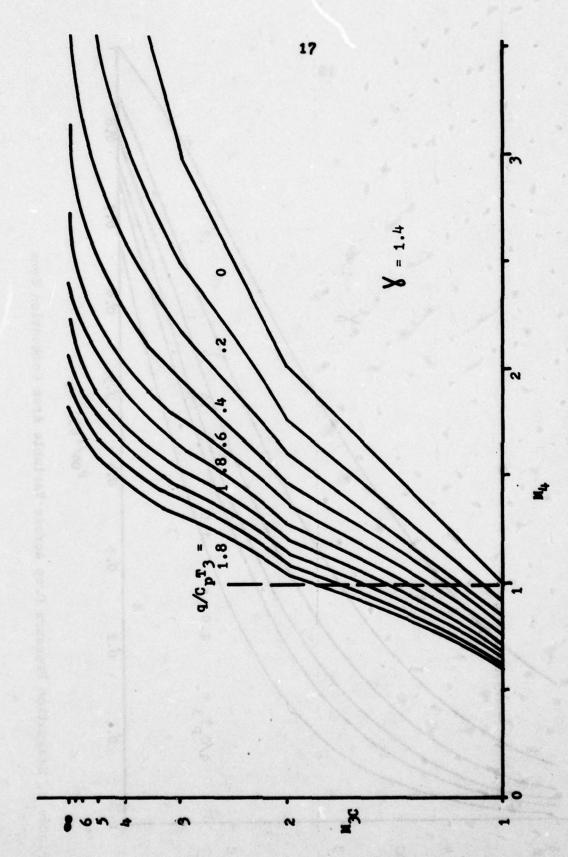


Figure 7. Supersonic Flow Required at Entrance to the Combustion Chamber as a Function of the Heat Release

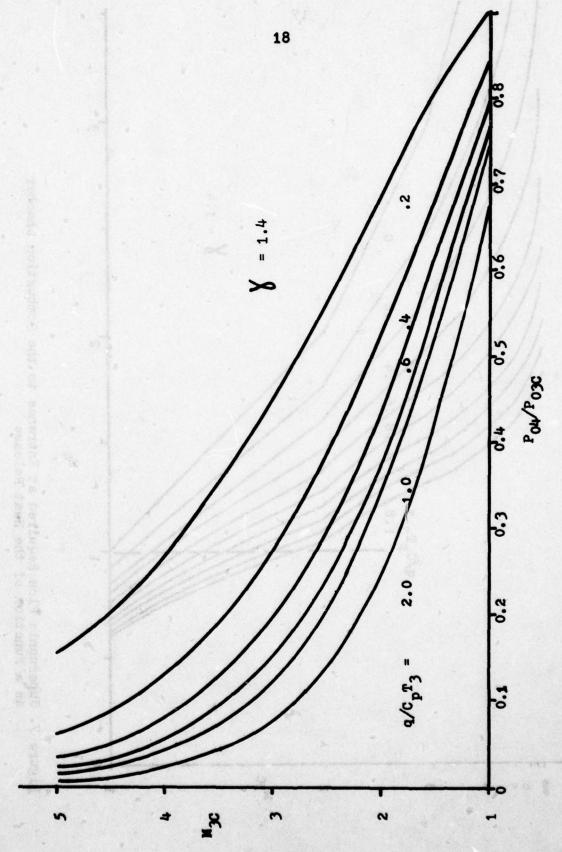


Figure 8. Stagnation Pressure Drop Across Variable Area Combustion Zone

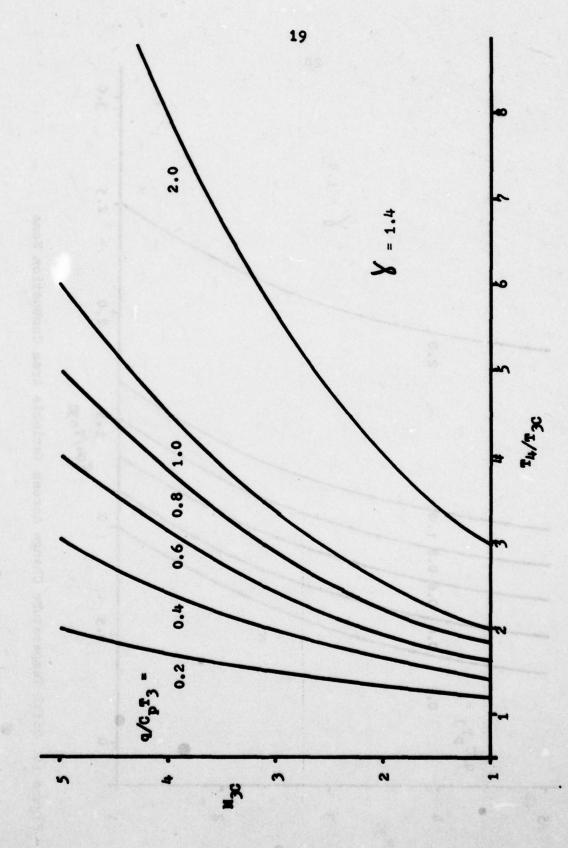


Figure 9. Temperature and Area Increase Across Variable Area Combustion Zone

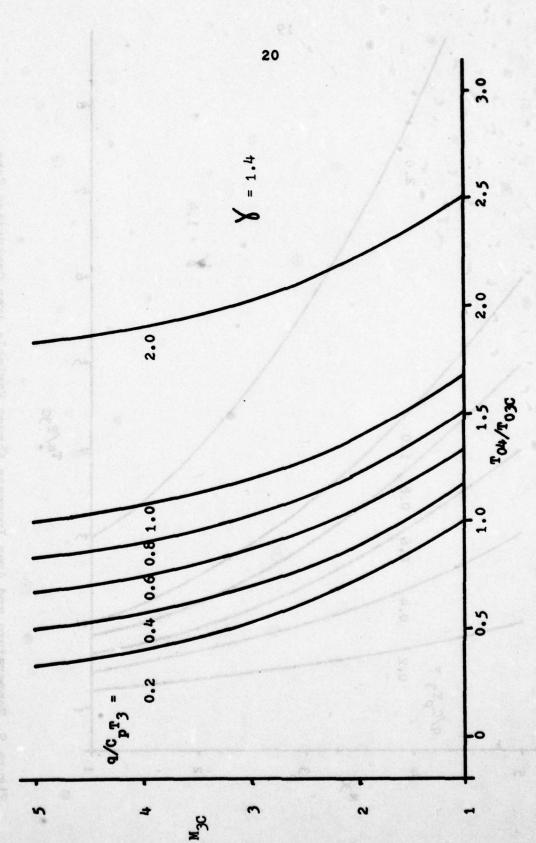


Figure 10. Total Temperature Change Across Variable Area Combustion Zone

with an appropriate increase in YFROZ. Inspection of Figure 7 shows that for constant values of heat release coefficient at the combustion chamber exit, larger mach numbers are required at the diffuser exit to lower the resultant mach numbers at the combustion chamber exit.

#### SECTION VII

SUPERSONIC AIR FLOW AND HYDROGEN FUEL COMBUSTION CALCULATIONS

It was determined for the lower speeds of  $M_{\infty} = 5$  and  $O_0^{\circ} = 0.5$  that the temperature reduced to 739 °K for the shocked inlet and 754.9°K for the shock free inlet. For these calculations from then on out to the nozzle exit it was assumed that ignition of the hydrogen-air mixture was achieved by other means rather than self ignition as the flow would quench below approximately 900 °K. The next procedure is to calculate the fluid properties of the hydrogen-air mixture at the combustion entrance and exit for chemically reacting gases. Consider the energy equation,

$$h_{fabs}^{3c} + \frac{u_{3c}^{2}}{2} = h_{fabs}^{4} + \frac{u_{4}^{2}}{2}$$

The assumption of constant pressure combustion implies that dp=0 and therefore du=0 from the momentum equation.

That is, the enthalpy of the hydrogen-air mixture before combustion is equal to the enthalpy of the combustion of gases at the combustion chamber exit.

Recalling that it is assumed that the temperature of the hydrogen entering the airstream is 1000 °K or less (equivalent to the temperature of the airstream.) For example, if the airstream was 3000 °K, the hydrogen temperature is 1000 °K, and if the temperature of the airstream were 950 °K, the hydrogen temperature is given as 950 °K also. The reason for this was to avoid the possibility of dissociation

of the H<sub>2</sub> molecules into H atoms before they enter the airstream. This would affect the enthalpy, molecular weight of the mixture. Even though this effect would be slight the enthalpic state of hydrogen is much larger than air because of its much smaller molecular weight of 2.016 versus 28.853 for air. This could mean changes of up to 100 °K in the final computations, and it is also unlikely that H<sub>2</sub> would be injected into the airstream at higher temperatures.

Now to determine the temperature at the exit of the combustion chamber, values are actually just guessed at approximately 1500-2000 OK above that of the air at the entrance of the combustion chamber. The pressure of course was assumed to be constant.

The first thing to do is to calculate the mole fractions at a given pressure and temperature.

$$\mathcal{R}_o = K^{(\frac{1}{2}O_2)} \sqrt{\mathcal{R}_{o_2}}$$

$$\mathcal{N}_{H_2} = \left\{ \sqrt{\left(\frac{A}{B}\right)^2 + \frac{1 - 0.5 \, \mathcal{N}_0}{B}} - \frac{A}{B} \right\}^2$$

where

$$A = 0.25 \left[ K^{\left(\frac{1}{2} H_2\right)} + \left( \frac{n_{o_2}^9 + n_{N_2}^9}{n_{H_2}^9} + 1 \right) K^{\left(\frac{1}{2} H_2\right)} + N_3^{(oH)} \right]$$
and

$$B = 0.5 + \left(\frac{n_{0_2}^3 + n_{N_2}^3}{n_{N_2}^3} + 0.5\right) + K^{(N_20)} \sqrt{n_{0_2}^3}$$

$$\eta_{H} = K^{(\frac{1}{2}H_{2})} \sqrt{\eta_{H_{2}}}$$

$$\eta_{OH} = K^{(OH)} \sqrt{\eta_{O_{2}}} \sqrt{\eta_{H_{2}}}$$

$$\eta_{H_{2}O} = K^{(H_{2}O)} \sqrt{\eta_{O_{2}}} \eta_{H_{2}}$$

$$\eta_{N_{2}} = \left\{ \sqrt{\left[0.25 K^{(NO)} \sqrt{\eta_{O_{2}}}\right]^{2} + \left(\eta_{H_{2}O} + \eta_{H_{2}} + 0.5 \left[\eta_{OH} + \eta_{H_{1}}\right] \frac{\eta_{N_{2}}^{3}}{\eta_{N_{2}}^{3}} - 0.25 K^{(NO)} \sqrt{\eta_{O_{2}}} \right\}^{2}}$$
and
$$\eta_{NO} = K^{(NO)} \sqrt{\eta_{O_{2}}} \sqrt{\eta_{N_{2}}}$$

These calculations are repeated with improved values of the estimated mole fractions of oxygen until

$$\left|\frac{n_{o_2}^9}{n_{H_2}^9} - \mathcal{J}_{o_2}\right| < .001 \frac{n_{o_2}^9}{n_{H_2}^9} = \frac{n_{o_2} + 0.5(n_0 + n_{H_20} + n_{OH} + n_{H_2})}{n_{H_20} + n_{H_2} + 0.5(n_{OH} + n_{H_2})}$$

Successively closer values of the enthalpic state at the exit of the combustion are determined until an accuracy of

$$\left| \frac{h_{f \text{ mix}} - h_{f \text{ mix}}^{T_g}}{h_{f \text{ mix}}^{T_g}} \right| \leq 0.01 \text{ is achieved.}$$

Since again the mach number can be calculated in the same manner as at the nozzle throat. Viewing the results of the calculations, it is evident that water and nitrogen are the final chemical constituients provided the temperatures are not to great and the hydrogen, oxygen and nitrogen atoms are not allowed to reform into molecules.

## SECTION VIII

## ISENTROPIC FLOW CALCULATIONS AT NOZZLE

The most desirable or ideal state at the exit of the engine would be that no underexpanded flow,  $P_{\text{exit}} < P_1$  or overexpanded flow where  $P_{\text{exit}} > P_1$  would exit the end of the nozzle. For the purpose of calculating the ideal nozzle flow the pressure was assumed equal to that entering the engine behind the bow shock wave.  $P_5 = 0.02$  for  $M_1 = 5$  and  $P_5 = 0.04$  for  $M_1 = 8$ . Also it was assumed that the flow went from the combustion chamber exit to the nozzle exit isentropically. A reasonably valid assumption provided the hydrogen gas combustion reaction is complete. That is, the hydrogen gas has released all its heat content within the combustion chamber and no additional heat is released within the nozzle exit. The decrease in temperature and resulting increase in velocity of the gases is a transformation of molecular random motion into more ordered and directional motion.

The entropy at the exit of the combustion chamber can be determined as the temperature, pressure and mole fractions are known.

$$\left(\frac{s}{R_2}\right)^{T_4} = \frac{m_2}{m_4} \left[ \sum_i \eta_{i,4} \left(\frac{s}{R}\right)^{T_4} - \sum_i \eta_{i,4} - \ln P_4 \right]$$

Pressure and temperature will decrese for nozzle expansion and the velocity will increase. The frozen flow mach number and are calculated at station 4 and using these values the estimates at station 5 are made with perfect gas isentropic relationships.

$$\frac{P_{5}}{P_{05}}\Big|_{M_{5}} = \frac{P_{5}}{P_{4}} \frac{P_{4}}{P_{04}}\Big|_{M_{4}}$$
where
$$\frac{P_{04}}{P_{4}} = \left[1 + \frac{y-1}{2} M_{4}^{2}\right]^{\frac{y}{y}-1}$$

$$M_{5} = \left\{\left(\frac{P_{05}}{P_{5}}\right)^{y-1/y} - 1\right\} \frac{2}{y-1}\right\}^{1/2}$$

After estimating M<sub>5</sub> the temperature corresponding to the pressure is determined.

$$T_{5} = \frac{T_{5}}{T_{05}} \frac{T_{04}}{T_{4}} T_{4} = \frac{\left[1 + \frac{y-1}{2} M_{4}^{2}\right]}{\left[1 + \frac{y-1}{2} M_{5}^{2}\right]} T_{4}$$

Keeping in mind that the value of \( \) used is that frozen from the combustion chamber exit. The temperature and pressure are adjusted accounting for chemical reactions and with this estimate of temperature the mole fractions from the previous equations given are calculated. The entropy is.

$$\left(\frac{s}{R_2}\right)^{T_5} = \frac{m_2}{m_4} \left[ \sum_{i} n_{i,4} \left(\frac{s}{R}\right)_{i}^{T_5} - \sum_{i} n_{i,5} \ln n_{i,5} - \ln p \right]$$

The temperature  $T_5$  is adjusted until the entropy at station 5 is within 0.01% of that at station 4. The corresponding absolute enthalpy is then determined.

$$\frac{h_{fabs}}{R_{z}T_{z}^{2}} = \frac{m_{z}}{m_{s}} \frac{T_{s}}{T_{z}^{2}} \sum_{i} \mathcal{R}_{i,s} \left( \frac{\Delta H_{fabs}}{\mathcal{R}T} \right)^{T_{s}}$$

$$\mathcal{U}_{s}^{cauc} = \left\{ \left[ \left( \frac{h_{fabs}}{R_{a}} \right)^{T_{s}} - \left( \frac{h_{fabs}}{R_{a}} \right)^{T_{s}} \right] \frac{2 \left( 83/4.35 \right) T_{s}}{m_{s}} + \mathcal{U}_{s} \right\}$$

$$\mathcal{V}_{s}^{fROZ} = \frac{\sum_{i} \mathcal{R}_{i,s} \left( \frac{C_{s}}{R_{a}} \right)^{T_{s}}}{\sum_{i} \mathcal{R}_{i,s} \left( \frac{C_{s}}{R_{a}} \right)^{T_{s}}} - 1$$

$$\mathcal{M}_{s}^{fROZ} = \frac{\mathcal{U}_{s}^{cauc}}{\sqrt{\mathcal{V}_{s}^{fROZ}} \frac{2C_{s}}{m_{s}} T_{s}^{cauc}}$$

#### SECTION IX

CALCULATION OF AREA RATIOS FROM COMPUTED FLOW PROPERTIES

The area ratios can now be determined from the conservation of mass and the equation of state.

$$(\rho u A)_1 = (\rho u A)_2 \qquad \rho = \frac{P}{T} \frac{m}{R}$$
or 
$$\frac{A_2}{A_1} = \frac{T_2}{T_1} \frac{P_1}{P_2} \frac{m_1}{m_2} \frac{u_2}{u_2}$$

Note that a specific area cannot be found for any one location. A reference area must be established. The sonic throat is chosen at the reference area as it supposedly represents the smallest area. The area ratio at the throat is given and is that location where the flow is sonic (M = 1).

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## SECTION X

# SPECIFIC NET THRUST AND TSFC

The specific net thrust (SNT) and thrust specific fuel consumption (TSFC) are determined as follows:

$$\frac{T}{\dot{m}_a} = (1+f)U_5 - U_1 \qquad P_a = P_e$$

ma - mass flow rate of air through the engine configuration

f - fuel air ratio

u5 - exit velocity of burned gases

u1 - flight velocity of the engine (initially specified)

$$f = \frac{\text{mass fuel / per unit time}}{\text{mass air / per unit time}} = \frac{2.016 \frac{\text{Mg}}{\text{N-mole-unit time}}}{\sqrt[9]{32 + (3.76)(28.016)) \frac{\text{Mg}}{\text{N-mole-unit time}}}}$$

$$f = \frac{2.016}{0.5(32+3.76(28.016))} = 0.0293578 \text{ for } V_0^* = 1/2$$

$$f = \frac{2.016}{2(32+3.76(28.016))} = 0.0073394 \text{ for } V_0^* = 2$$

$$TSFC = \frac{m_f}{7} = \frac{f}{7} / \gamma_{m_g}$$

## SECTION XI

## CONCLUDING COMMENTS

The resulting specific net thrusts and associated TSFC are rather high. As would be expected the greatest amount of thrust would be provided for the richest mixture of hydrogen to air at  $\sqrt{0_2}^0 = 0.5$ . The thrust falls off dramatically for leaner mixtures and for the shocked inlet at  $M_{\infty} = 10$  for  $\sqrt{0_2}^0 = 2$  negative thrust or drag is provided.

Rather encouraging results are obtained for the shocked inlet. The thrust values are reasonably high and the associated TSFC's are almost double that of some typical ramjets (designed and in operation.) This type of design would require that the stagnation pressure drop such that  $P_{3C}/P_{03C}$ ,  $P_{4}/P_{04}$ , and  $P_{5}/P_{05}$  be below values of 0.528 or it would not be possible to obtain supersonic flow in the aft side of the diffuser. This can be determined from perfect gas tables, shock tables, and the graphs included herein for stagnation pressure drops across the combustion chamber.

For  $M_{\infty} = 5$ ,  $\sqrt{\frac{0}{2}}^{\circ} = 0.5$ , and the shocked inlet,  $P_{3C}/P_{03C} = 0.1113$   $P_{4}/P_{04} = 0.3969$   $P_{5}/P_{05} = 0.10994$ all of which are below  $P/P_{0} = 0.528$ . This means the flow will indeed go supersonic. Similarly, the other conditions can be checked.

TABLE 1 CALCULATED SNT AND TSFC VALUES

	INLET ENGINE	7/m <sub>a</sub> (N/Kg/sec)	TSFC (Kg/hr/N)
M <sub>co</sub>	Jo2°	//ma (N/Ng/Sec)	TSFC (Kg/hr/N)
5	0.5	341.334	0.3096
5	2	110.312	0.2395
10	0.5	441.404	0.2394
10	2 s'otal becaloomal	-359.567	NA WELLER
SHOCK F	REE ENGINE		
M <sub>∞</sub>	$\mathcal{J}_{0_2}{}^{\circ}$	$\mathcal{T}/m_{a}$ (N/Kg/sec	TSFC (Kg/hr/N)
5 ** ** *	0.5	901.037	0.1173
5	2	230.790	0.1145
10	0.5	739.172	0.1430
10	2	-12.412	NA NA
	8000 F. C =	Fa/2,0 = 0,3969 - 2,77	
SOME TY	PICAL VALUES OF TS	FC FOR OTHER TYPES OF ENGIN	VES ARE

		TO CELEBRATE OF	DIGINDO AIG
RAMJET	0.173 - 0.265	(N/Kg/sec)	M = 2
TURBOJETS	0.0763 - 0.1078	(N/Kg/sec)	STATIC
TURBOFANS	0.0509 - 0.0611	(N/Kg/sec)	STATIC

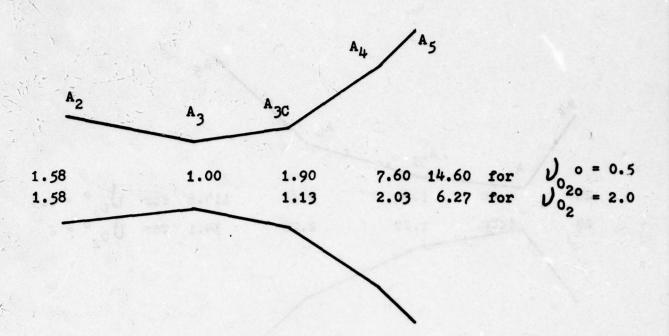


Figure 11. Shocked Inlet for  $M_{\infty} = 5$ ,  $J_{0_2}^{\circ}$  0.5, 2.0 with Area Relationships

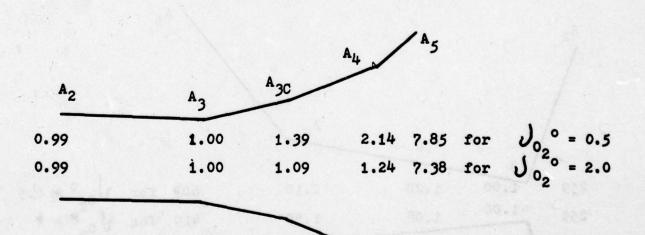


Figure 12. Shocked Inlet for Me = 10,  $v_0^0 = 0.5$ , 2.0 with Area Relationships

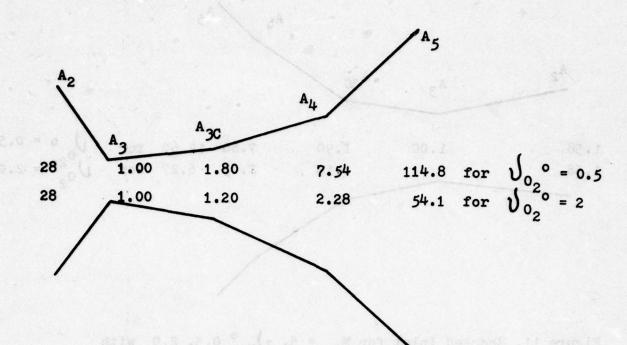
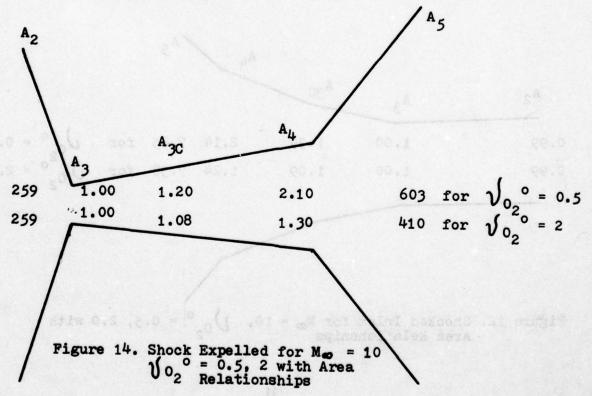


Figure 13. Shock Expelled for  $M_{\infty} = 5$ ,  $\sqrt{0_2}^{\circ} = 0.5$ , 2 with Area Relationships



While the values of thrust and TSFC for the shock free engine look to be promising, it must also be considered that the static pressures of up to 272 atmospheres would be prohibitive. In fact, it would be necessary to open up the diffuser rather than closing it to provide permissible operating pressures. Also, to prevent choking in the combustion chamber the engine would have to be opened even more. This would require a tremendously large engine with a large associated wave drag. As such, only the shocked inlet configuration appears to be promising.

A method of injecting and combusting the gases across oblique shock waves in the combustion chamber for the shocked inlet configuration should be investigated as to its feasibility. Also, the same methods outlined in this paper could be used to investigate the feasibility of an external ramp over which combustion would take place on the aft side.

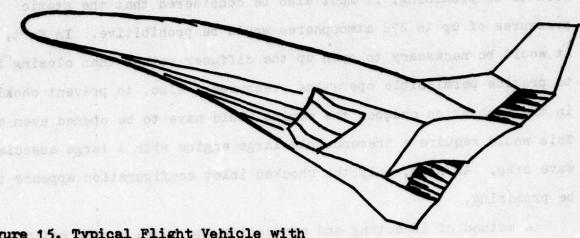


Figure 15. Typical Flight Vehicle with Supersonic Ramjet Engine

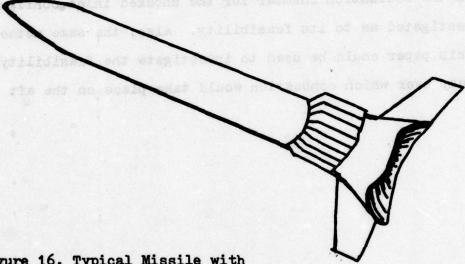


Figure 16. Typical Missile with Supersonic Ramjet Engine

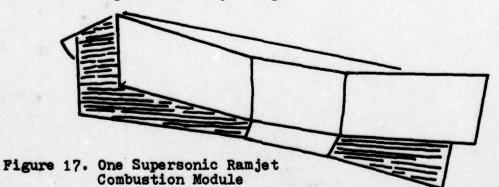


TABLE 2 COEFFICIENTS a (j)(T) OF EQUILIBRIUM CONSTANTS Kp (j)(T)

1100 11.036 39.70% 6.476 0.114; 1201 11.036 39.70% 6.30% 0.10%; 1301 11.619 40.70 6.30% 0.10%; 1401 11.653 40.101 6.29% 0.10%; 1500 15.049 40.001 6.8436 0.00%; 1600 15.752 41.033 6.1933 0.09%; 1700 15.426 41.244 6.1007 0.00%; 1800 15.757 41.436 6.00% 0.00%; 1800 15.757 41.436 6.00% 0.00%; 1800 15.753 41.460 6.010% 0.00%; 1800 15.753 41.460 6.010% 0.00%; 1800 15.753 41.460 6.00% 0.00%; 1800 16.119 41.47% 5.9440 0.00%; 1800 16.211 41.436 5.9710 0.00%; 1800 16.211 41.436 5.9710 0.00%; 1800 16.255 41.35% 5.9703 0.00%; 1800 16.255 41.35% 5.9703 0.00%; 1800 16.351 41.269 5.8457 0.00%; 1800 16.451 41.112 5.7610 0.00%; 1800 16.451 41.112 5.7610 0.00%; 1800 16.543 40.862 5.7171 0.03%; 1800 16.543 40.862 5.7171 0.03%; 1800 16.602 40.531 5.694 0.083; 1800 16.602 40.531 5.694 0.083; 1800 16.602 40.531 5.694 0.083; 1800 16.602 40.531 5.694 0.083; 1800 16.602 40.531 5.580 0.083; 1800 16.642 40.661 5.580 0.088; 1800 16.645 39.931 5.5940 0.088; 1800 16.645 39.931 5.5940 0.088; 1800 16.645 39.931 5.5940 0.088; 1800 16.645 39.931 5.5940 0.088; 1800 16.645 39.931 5.5940 0.088; 1800 16.645 39.931 5.5940 0.088; 1800 16.645 39.931 5.5940 0.088; 1800 16.645 39.931 5.5940 0.088; 1800 16.645 39.931 5.5940 0.088; 1800 16.645 39.931 5.5940 0.088; 1800 16.645 39.931 5.5940 0.088; 1800 16.645 39.931 5.5940 0.088; 1800 16.655 39.433 5.8604 0.0828;
1400 14.653 40.401 6.2915 0.0018 1500 15.049 40.401 6.2436 0.0018 1600 15.752 41.013 6.193 0.0799 1700 15.458 41.244 6.1493 0.0739 1800 15.02 41.3.4 6.1007 0.0231 1900 15.757 41.438 6.0564 0.0025 2000 15.653 41.460 6.0105 0.0152 2100 16.002 41.479 5.9440 0.0871 2200 16.119 41.479 5.9440 0.0871 2300 16.211 41.434 5.9110 0.0861 2400 16.275 41.374 5.9703 0.0053 2400 16.351 41.269 5.857 0.0053 2500 16.551 41.207 5.8133 0.0053 2600 16.451 41.112 5.7810 0.0840 2700 16.553 40.993 5.7490 0.0840 2800 16.553 40.852 5.7171 0.8376 2800 16.553 40.852 5.7171 0.08376 2800 16.562 40.553 5.6594 0.0838 2800 16.662 40.553 5.6594 0.0838 2800 16.665 30.901 5.6594 0.0838 2800 16.665 39.901 5.5564 0.0838 2800 16.665 39.901 5.5564 0.0838 2800 16.665 39.901 5.5564 0.0838
1400 14.653 40.401 6.2915 0.0018 1500 15.049 40.401 6.2436 0.0018 1600 15.752 41.013 6.193 0.0799 1700 15.458 41.244 6.1493 0.0739 1800 15.02 41.3.4 6.1007 0.0231 1900 15.757 41.438 6.0564 0.0025 2000 15.653 41.460 6.0105 0.0152 2100 16.002 41.479 5.9440 0.0871 2200 16.119 41.479 5.9440 0.0871 2300 16.211 41.434 5.9110 0.0861 2400 16.275 41.374 5.9703 0.0053 2400 16.351 41.269 5.857 0.0053 2500 16.551 41.207 5.8133 0.0053 2600 16.451 41.112 5.7810 0.0840 2700 16.553 40.993 5.7490 0.0840 2800 16.553 40.852 5.7171 0.8376 2800 16.553 40.852 5.7171 0.08376 2800 16.562 40.553 5.6594 0.0838 2800 16.662 40.553 5.6594 0.0838 2800 16.665 30.901 5.6594 0.0838 2800 16.665 39.901 5.5564 0.0838 2800 16.665 39.901 5.5564 0.0838 2800 16.665 39.901 5.5564 0.0838
1500 15.049 40.041 6.2436 0.00000 1600 15.752 41.0.3 6.193 0.09999 1700 15.428 41.244 6.1007 0.00000 1500 15.757 41.434 6.1007 0.00000 15.653 41.400 6.0105 0.00000 2000 15.653 41.400 6.0105 0.00000 2000 15.653 41.400 6.0105 0.00000 2000 16.119 41.477 5.9440 0.00000 2000 16.211 41.434 5.9110 0.00000 2000 16.211 41.434 5.9110 0.00000 2000 16.255 41.354 5.8783 0.00000 2000 16.351 41.269 5.8657 0.00530 2000 16.451 41.126 5.7810 0.00000 2000 16.451 41.112 5.7810 0.00000 2000 16.543 40.050 5.7490 0.00000 2000 16.553 40.050 5.7490 0.00000 2000 16.553 40.050 5.7490 0.00000 2000 16.553 40.050 5.7490 0.00000 2000 16.555 40.719 5.6654 0.00330 2000 16.600 40.553 5.6594 0.00330 2000 16.600 40.553 5.6594 0.00330 2000 16.600 40.553 5.6594 0.00330 2000 16.600 40.553 5.6594 0.00330 2000 16.600 40.553 5.6594 0.00330 2000 16.600 40.553 5.6594 0.00330 2000 16.600 40.553 5.6594 0.00330 2000 16.600 40.553 5.6594 0.00330 2000 16.600 40.553 5.6594 0.00330 2000 16.600 40.553 5.6594 0.00330 2000 16.600 40.553 5.6594 0.00330 2000 16.600 40.553 5.6594 0.00330 2000 16.600 40.553 5.5564 0.00330 2000 16.600 40.553 5.5564 0.00330 2000 16.600 40.553 3.5564 0.00330 2000 16.600 40.553 3.5564 0.00330 2000 16.600 40.553 3.5564 0.00330
1700 15.428 41.244 6.1007 0.0132 1800 15.427 41.345 6.1007 0.0133 1800 15.427 41.345 6.0574 0.0133 1800 15.727 41.345 6.0574 0.0133 1800 15.553 41.460 6.0105 0.0136 1800 16.002 41.422 5.9771 0.0681 1800 16.119 41.477 5.9440 0.05711 1800 16.211 41.434 5.9110 0.05711 1800 16.275 41.354 5.8783 0.00241 1800 16.351 41.269 5.8457 0.00531 1860 16.405 41.207 5.8133 0.00483 1860 16.451 41.112 5.7810 0.08483 1800 16.451 41.112 5.7810 0.08483 1800 16.553 40.862 5.7171 0.0837 1800 16.543 40.862 5.7171 0.0837 1800 16.555 40.719 5.6654 0.08383 1800 16.620 40.553 5.6594 0.08383 1800 16.620 40.391 5.6934 0.08383 1800 16.620 40.391 5.6934 0.08383 1800 16.620 40.391 5.6934 0.08383 1800 16.635 40.225 5.6076 0.08383 1800 16.645 39.901 5.5564 0.08283
1800 15.797 11.838 6.0574 0.0273 1800 15.797 11.838 6.0574 0.0037 1800 15.653 11.840 6.0105 0.0175 1800 15.653 11.840 6.0105 0.0175 1800 16.119 11.877 5.9840 0.0571 1800 16.211 11.834 5.9110 0.0854 1800 16.275 11.354 5.8783 0.0058 1800 16.351 11.879 5.857 0.0058 1800 16.451 11.127 5.8133 0.0088 1800 16.451 11.112 5.7810 0.0846 1800 16.451 11.112 5.7810 0.0846 1800 16.543 10.852 5.7171 0.0837 1800 16.543 10.852 5.7171 0.0837 1800 16.555 10.719 5.6854 0.0338 1800 16.562 10.933 5.6594 0.0338 1800 16.662 10.931 5.6834 0.0838 1800 16.662 10.931 5.6334 0.0838 1800 16.662 10.931 5.6334 0.0838 1800 16.662 10.931 5.6334 0.0838 1800 16.662 10.931 5.6334 0.0838 1800 16.662 10.931 5.6334 0.0838
1900 15.757 \$1.898 6.0564 0.0035 2000 15.653 \$1.860 6.0105 0.0036 2100 16.002 \$1.862 5.9771 0.0681 2200 16.119 \$1.879 5.9880 0.0571 2300 16.211 \$1.834 5.9110 0.0654 2800 16.275 \$1.344 5.9783 0.0056 2800 16.351 \$1.269 5.8857 0.0053 2800 16.451 \$1.120 5.8133 0.0056 2700 16.851 \$1.112 5.7810 0.0886 2800 16.851 \$1.112 5.7810 0.0886 2800 16.851 \$1.112 5.7810 0.0886 2800 16.853 \$0.852 5.7171 0.0837 3000 16.565 \$0.719 5.6854 0.0838 3100 16.602 \$0.553 5.6594 0.0838 3200 16.602 \$0.553 5.6594 0.0838 3200 16.602 \$0.553 5.6594 0.0838 3200 16.602 \$0.553 5.6596 0.0838 3200 16.602 \$0.553 5.6596 0.0838 3200 16.605 \$0.225 5.6076 0.0828 3300 16.635 \$0.225 5.6076 0.0828 3300 16.645 39.901 5.5564 0.0828
2000 15.653 41.460 6.0105 0.00562 2100 16.002 41.462 5.47/1 0.08811 2200 16.119 41.47, 5.9440 0.057/1 2300 16.211 41.434 5.9110 0.065/1 2400 16.255 41.354 5.8783 0.005/2 2500 16.351 41.269 5.8657 0.00531 2600 16.451 41.126 5.7810 0.08440 2700 16.451 41.112 5.7810 0.08440 2800 16.453 40.052 5.71/1 0.083/2 2500 16.543 40.052 5.71/1 0.083/2 2500 16.543 40.052 5.71/1 0.083/2 2500 16.543 40.052 5.71/1 0.083/2 2500 16.543 40.052 5.71/1 0.083/2 2500 16.662 40.553 5.6594 0.083/2 2500 16.602 40.553 5.6594 0.083/2 2500 16.602 40.591 5.6954 0.083/2 2500 16.602 40.591 5.6954 0.083/2 2500 16.602 40.591 5.5964 0.083/2 2500 16.605 39.901 5.5564 0.082/2 2500 16.645 39.901 5.5564 0.082/2
2000 16.119 kg.kyy 5.9kho 0.05/11 2500 16.211 kg.kyi 5.9110 0.06/11 2500 16.275 kg.yiky 5.9110 0.06/11 2500 16.351 kg.ziky 5.8kyy 0.005/31 2500 16.351 kg.ziyy 5.8kyy 0.005/31 2500 16.451 kg.liz 5.7810 0.08kyi 2500 16.451 kg.liz 5.7810 0.08kyi 2500 16.451 kg.liz 5.7810 0.08kyi 2500 16.5kyi ko.993 5.7490 0.08kyi 2500 16.5kyi ko.993 5.669k 0.0838i 2500 16.620 ko.953 5.659k 0.0838i 2500 16.635 ko.225 5.6076 0.0828i 2500 16.645 39.901 5.556k 0.0828i 2500 16.645 39.901 5.556k 0.0828i
2900 16.211 11.434 5.9110 0.0854 2800 16.275 11.354 5.8783 0.0058 2800 16.351 11.289 5.857 0.0058 2800 16.451 11.112 5.7810 0.0088 2800 16.451 11.112 5.7810 0.0088 2800 16.451 11.112 5.7810 0.0848 2800 16.453 10.852 5.7171 0.0837 2800 16.543 10.852 5.7171 0.0837 2800 16.555 10.719 5.6854 0.033 2800 16.602 10.391 5.6854 0.033 2800 16.602 10.391 5.6334 0.0832 2800 16.602 10.391 5.6334 0.0833 2800 16.602 10.391 5.6334 0.0833 2800 16.602 10.391 5.6334 0.0833 2800 16.603 10.255 5.6076 0.0833 2800 16.605 39.901 5.5564 0.0838
2400 16.275 k1.35k 5.6783 0.00588 2500 16.351 k1.289 5.8k57 0.00531 2600 16.k55 k1.207 5.8133 0.00883 2700 16.k51 k1.112 5.7810 0.0880 2800 16.k55 k0.312 5.7810 0.0880 2900 16.5k3 k0.862 5.7171 0.0837 2000 16.5k3 k0.862 5.7171 0.0837 2000 16.505 k0.719 5.685k 0.00358 2100 16.602 k0.553 5.659k 0.00388 2100 16.602 k0.553 5.659k 0.00388 2100 16.602 k0.553 5.659k 0.00388 2100 16.602 k0.503 5.6058 0.00388 2100 16.602 k0.503 5.6058 0.00388 2100 16.602 k0.391 5.6038 0.00388 2100 16.605 k0.225 5.6076 0.00288 2100 16.605 39.901 5.556k 0.00288
16.351
700 16.451 41.112 5.7810 0.0846 8600 16.451 40.93 5.7810 0.0846 9600 16.543 40.852 5.7171 0.0837 9000 16.543 40.852 5.7171 0.0837 9000 16.562 40.933 5.6854 0.0838 9000 16.620 40.931 5.6334 0.0838 9000 16.620 40.391 5.6334 0.0830 9000 16.652 40.025 5.6076 0.0838 9000 16.652 40.001 5.9820 0.0838 9000 16.655 39.901 5.5564 0.0828 9000 16.655 39.901 5.5564 0.0828 9600 16.655 39.901 5.5564 0.0828 9600 16.655 39.901 5.5564 0.0828 9600 16.655 39.901 5.5564 0.0828 9600 16.655 39.901 5.5564 0.0828 9600 16.655 39.901 5.5564 0.0828 9600 16.655 39.901 5.5564 0.0828 9600 16.655 39.901 5.5564 0.0828 9600 16.655 39.901 5.5564 0.0828 9600 16.655 39.901 5.5564 0.0828 9600 16.655 39.901 5.5564 0.0828 9600 16.655 39.738 5.5310 0.0828 9600 16.655 39.738 5.5310 0.0828 9600 16.655 39.738 5.5310 0.0828 9600 16.655 39.738 5.5310 0.0828 9600 16.655 39.738 5.5310 0.0828 9600 16.655 39.738 5.5310 0.0828 9600 16.655 39.738 5.5310 0.0828 9600 16.655 39.738 5.5310 0.0828 9600 16.655 39.738 5.5310 0.0828 9600 16.655 39.738 5.5310 0.0828 9600 16.655 39.738 5.5310 0.0828 9600 16.655 39.738 5.5310 0.0828 9600 16.655 39.738 5.5310 0.0828 9600 16.655 39.738 5.5310 0.0828 9600 16.655 39.738 9600 16.655 9600 16.655 9600 16.655 9600 16.655 9600 16.655 9600 16.655 9600
9000 16.543 40.852 5.7171 0.08370 9000 15.595 40.719 5.6854 0.00352 9100 16.602 40.553 5.6594 0.08382 9200 16.620 40.391 5.6334 0.08387 9100 16.635 40.225 5.6076 0.08297 9100 16.642 40.061 5.9820 0.08387 9100 16.645 39.901 5.9564 0.08287 9100 16.645 39.901 5.9564 0.08287
2900 16.5\(\frac{1}{3}\) \(\lambda \).\(\frac{1}{3}\) \(\frac{1}{3}\) \(\lambda \).\(\frac{1}{3}\) \(\frac{1}{3}\) \(\frac{1}\) \(\frac{1}{3}\) \(\frac{1}\) \(\frac{1}{3}\) \(\frac{1}\) \(\frac{1}\) \(\frac{1}{3}\) \(\frac{1}\) \(
3000 1.5.585 k0.719 5.685k 0.0035i 3100 16.602 k0.553 5.659k 0.003si 3200 16.620 k0.391 5.633k 0.0835i 3300 16.635 k0.255 5.6076 0.0025i 3400 16.645 k0.001 5.5820 0.0025i 3500 16.645 39.901 5.556k 0.0025i 3600 16.645 39.738 5.5310 0.0025i
3000 16.620 40.391 5.6394 0.06397 3300 16.635 40.225 5.6076 0.00299 3400 16.642 40.001 5.5820 0.00299 3500 16.645 39.901 5.5564 0.00288
3300 16.635 40.225 5.6076 0.0029 3400 16.642 40.001 5.5820 0.0026 3300 16.645 39.901 5.5564 0.0026 3600 16.645 39.738 5.5310 0.0026
3600 16.642 kg.(s)1 5.5826 g.62685 3500 16.645 39.901 5.5564 g.62685 3600 16.645 39.736 5.5310 g.6265
3500 16.645 39.901 5.9564 6.00000 3600 16.645 39.738 5.5310 6.00000
3700 16.638 39.575 5.5056 0.08e8
900 16.625 39.413 5.4804 0.0828 900 16.612 39.20 5.4553 0.0829
9900 16.612 39.2% 5.4553 0.08291 0000 16.602 39.087 5.4303 0.083901
A100 16.588 38.90A 5.A081 0.08301
1900 16.557 38.544 5.3539 0.06331 1400 16.539 38.363 5.3480 0.06388
300 16.504 38.225 5.3201 0.08333
600 16.479 38.046 5.8984 0.08348 700 16.453 37.868 5.8767 0.08358
\$700 16.453 37.868 5.2767 0.08358 \$800 16.423 37.690 5.2551 0.08358
900 16.392 37.313 5.2122 0.08376 000 16.359 37.334 5.2122 0.08376
31.00 16.325 37.159 5.1909 0.08383
\$200 16.289 36.985 5.1636 0.0332 5300 16.240 36.011 5.1485 0.08463
900 16.163 36.463 5.1064 0.08426
900 16.163 36.463 5.10CA 0.0BARG
9600 16.122 36.242 5.0055 0.0My
5700 16.079 36.118 5.0647 0.06447 5800 16.035 35.949 5.0440 0.09458
5000 16.035 35.949 5.0440 0.09454 5000 15.591 35.780 5.0834 0.09474
900 15.991 35.700 5.6234 0.64477 000 15.943 35.613 5.600H 0.69490
(a) 259/2 296h5 4615 -2013;
(3) 0.5 0.5 0.0 -0.
$(3)(\pi) = e^{(1)}(\pi_1) \left[ \frac{e^{(1)}(\pi_1 + 100)}{e^{(1)}(\pi_2)} \right]^{\frac{1}{100}} \cdot \pi^{(3)} \cdot ent \left[ -(\Lambda \chi_1^{(1)}) \Lambda \chi_2^{(1)} \right] \cdot (1/\pi)$

TABLE 3 COEFFICIENTS  $a^{(j)}(T_1)$  OF EQUILIBRIUM CONSTANTS  $K_p^{(j)}(T_1)$ 

(x)	(M <sub>2</sub> )	ju <sub>0</sub> + ½1+ + m) (m)	(O)
1100 1200 1300 1400 1500	35.800 37.353 37.909 38.272 38.555	4.2710 4.2902 4.3215 4.3432 4.3623	1.7175 -5 1.6586 -5 1.0126 -5 1.5787 -5 1.5537 -5
1600 1700 1800 1900	38.873 39.122 39.322 39.444 39.523	h.3/93 h.3934 h.4079 h.4194 h.4256	1.5341 -5 1.5241 -5 1.5127 -5 1.5048 -5 1.4985 -5
2100 2300 2400 2500	39.674 39.765 39.813 39.845 39.866	4.4393 4.4460 4.4531 4.4576 4.4680	1.4977 -5 1.4988 -5 1.4987 -5 1.5082 -5 1.5050 -5
2600 2700 2800 2900 3000	39.865 39.861 39.865 39.809 39.734	4.4691 h.4669 h.4687 h.4690 h.4709	1.5137 -5 1.5199 -5 1.5236 -5 1.5366 -5
3100 3200 3400 3500	39.701 39.666 39.630 39.986 39.539	4. 4709 4. 4709 4. 469 4. 4669	1.5441 -3 1.5526 -5 1.5615 -5 1.5703 -5 1.5786 -5
3600 3700 3600 3900	39.466 39.466 39.369 39.300 39.800	4.4647 4.4680 4.4564 4.4549 4.4513	1.5870 -5 1.5956 -5 3.6060 -5 1.6128 -5 1.6214 -5
#300 #300 #300 #300	39.157 39.063 39.013 36.990 36.866	4.4509 4.4380 4.4382 4.4256 4.4194	1.6297 -5 1.6383 -5 1.6470 -5 1.6570 -5 1.6659 -5
1700 1700 1800 1900	38.830 36.775 36.721 36.675 36.666	4.413? 4.4070 4.4009 4.3973 4.3877	1.6746 -5 1.6835 -5 1.6922 -5 1.7014 -5 1.7101 -5
3100 3200 3300 3400	36.582 36.536 36.501 36.463 36.488	4.3815 4.3750 4.3684 4.3514	1.7191 -5 1.7280 -5 1.7371 -5 1.7461 -5 1.7592 -5
9600 5700 9800 5900 6000	36.374 36.351 36.332 36.327	4.3475 4.3405 4.3395 4.3867 4.3198	1.7% -5 1.7720 -5 1.7810 -5 1.7900 -5 1.7905 -5
*) <sup>(3)</sup>	96613	10799	-33570
1)	0.5	0.0	0.125

TABLE 4 EQUILIBRIUM CONSTANTS, K(j)atm (j)

(K)	Kg - H	Epical elek	Hart Bos on	Kp (N-0) atm-6
1100	2.5704 -8	2.47/4 -9	9.1633 -2	7.6384 8
1200	1.9634 -7	2.4869 -8	1.27/2 -1	7.9250 7
1300	1.1015 -6	1.7579 -7	1.7378 -1.	1.158° 7
1400 1500	4.6417 -6	9.3972 -7	2.220 -1 2.7606 -1	2.8833 G
1600	5.4075 -5 1.4655 -4	1.4388 -5	3.3343 -1	1.5136 5
1700	3.5645 -4	4.43/1 -5	3.9PG4 -1	5.0003 4
1900	3.5645 -4 7.9068 -4	1.2078 -4 2.95du -4	5.1761 -1	7.6913 3
2000	1.6181 -3	6.6374 -4	5.8076 -1	3.4674 3
2100	3.1046 -3	1.3004 -3	6.4565 -1	1.08% 3
2200	5.6105 -3	2.6853 -3	7.0938 -1	8.7498 2
2300	9.6383 -3	4.9317 -3	7.7446 -1	4.60% 2 2.7733 2
2500	2.5061 -2	8.6097 -3 1.43AB -2	8.3753 -1 8.9750 -1	1.6749 2
2600	3.8194 -2	2.3121 -2	9.6161 -1	1.0495 2
2700	5.6624 -2	3.5810 -2	1.0233 0	6.80ri 1
2900 2900	8.1470 -2 1.1455 -1	5.3951 -2 7.8886 -2	1.0314 0	4.5499 1
3000	1.5740 -1	1.1246 -1	1.1967 0	3.1261 1 2.2029 1
3100	2.1184 -1	1.5668 -1	1.2531 0	1.5885 1
3200	2.7990 -1	2.1380 -1	1.300	1.1666 1
3300	3.6392 -1 4.6559 -1	2.86/12 -1 3.7757 -1	1.3583 0 1.4125 0	8.7498 o 6.6681 o
3500	5.8749 -1	4.8978 -1	1.4622 0	5.1523 0
3600	7.3282 -1	6.2517 -1	1.5101 0	4.0458 0
3700	9.0365 -1	7.8886 -1	1.5560 0	3.2137 0
3800	1.0990 0	9.8401 -1	1.6032 0	2,5882 0
900	1.5849 0	1.4791 0	1.6866 0	1.7298 0
100	1.8793 0	1.7865 0	1.7298 0	1.4355 0
1200	2.2080 0	2.1380 0	1.7701 0	1.1995 0
4300	2.5823 0	2.5351 0 2.9654 0	1.8072 0	1.0116 0 8.6099 -1
1500	2.9923 0 3.4435 0	3.4914 0	1.8836 0	7.3621 -1
600	3.9355 0	4.0551 0	1.9143 0	6.3533 ->
700	4.47/1 0	4.6881 0	1.9498 0	5.5081 -1
1900	5.0699 0 5.7148 0	5.3703 0 6.1376 0	1.9815 0	4.7973 -1
3000	6.3973 0	6.9663 0	2.0137 0	3.71% -1
5100	7.1450 0	7.8524 0	2.0749 0	3.2605 -1
200	7.9433 0	8.8308 0	2.1038 0	2.9942 -1
300	8.7902 0 9.6828 0	9.8855 0	2.1261 0 2.1528 0	2.6122 -1
5500	1.06A1 1	1.2246 1	2.1526 0 2.1827 0	2.3442 -1
5600	1.1641 1	1.3552 1	2.2029 0	1.9099 -1
5700	1.2706 1	1,4928 1	2,2284 0	1.72% -1
5000	1.3836 1	1.6406 1	2.2491 0	1.5740 -1
5900 5000	1.5031 1	1.7947 1	2.2751 0	1.3163 -1

TABLE 5 EQUILIBRIUM CONSTANTS, Kp (j) atm (j)

	416	Hills To the State of the State		
(K)	K <sup>(3)</sup> Mg) atm <sup>14</sup> Mg as H	K(NO) <sup>erm</sup> U	χ <sub>p</sub> (∞ <sub>2</sub> ) e te = ε	
1100	5.4325 -20	2.3281 -h	7.5858 8	
1200	4.1976 -18	5.3080 -h	5.8076 7	
1300	1.6672 -16	1.0666 -3	6.6222 6	
1400	3.9264 -15	1.9409 -3	1.0328 6	
1500	6.0674 -14	3.2584 -3	2.0701 5	
1600	6.6834 -13	5.1286 -3	5.0816 h	
1700	5.5463 -12	7.6560 -3	1.4791 h	
1800	3.6559 -11	1.0914 -2	4.9317 3	
1900	1.9724 -10	1.5031 -2	1.8493 3	
8000	8.9950 -10	1.9999 -2	7.6560 2	
2100	3.5563 -9	2.5942 -2	3.4594 2	
2200	1.2445 -8	3.2810 -2	1.6827 2	
2300	3.8994 -8	4.0644 -2	8.7056 1	
2400	1.1117 -7	4.9745 -2	4.7753 1	
2500	2.9174 -7	5.9293 -2	2.7542 1	
2600	7.0958 -7	7.0146 -2	1.6558 1	
2700	1.6218 -6	8.1846 -2	1.0351 1	
2800	3.4914 -6	9.4406 -2	6.6834 0	
2900	7.1285 -6	1.0789 -1	4.4566 0	
3000	1.3868 -5	1.2218 -1	3.0549 0	
3100	2.5882 -5	1.3709 -1	2.1478 0	
3200	4.6559 -5	1.5311 -1	1.5453 0	
3300	8.0724 -5	1.6943 -1	1.1324 0	
3400	1.3552 -4	1.8664 -1	8.4918 -1	
3500	2.2060 -4	2.0417 -1	6.4565 -1	
3600	3.5075 -4	2.2233 -1	4.9888 -1	
3700	5.4325 -4	2.4099 -1	5.9064 -1	
3600	8.2035 -4	2.6002 -1	3.1046 -1	
3900	1.2162 -3	2.7925 -1	2.4946 -1	
4600	1.7701 -3	2.9923 -1	2.0324 -1	
\$300 \$300 \$400 \$500	2.5235 -3 3.5481 -3 4.6978 -3 6.6681 -3 8.9743 -3	3.1915 -1 3.3884 -1 3.5975 -1 3.8019 -1 4.0087	1.6672 -1 1.3668 -1 1.1586 -1 9.7949 -2 8.3368 -2	
4600	1.1912 -2	4.2170 -1	7.1285 -2	
4700	1.5596 -2	4.8259 -1	6.1518 -2	
4800	2.0230 -2	4.6452 -1	5.3456 -2	
4900	2.6002 -2	4.8529 -1	4.6774 -2	
5000	3.3037 -2	5.0582 -1	4.1115 -2	
5300	4.1991 -2	5.2723 -1	3.6306 -2	
5800	5.2000 -2	5.4828 -1	3.8211 -2	
5300	6.4417 -2	5.6885 -1	2.8774 -2	
5400	7.9068 -2	5.9020 -1	2.5763 -2	
5500	9.6363 -2	6.1094 -1	2.3174 -2	
9600	1.1695 -1	6.3241 -1	2.0941 -2	
9700	1.4060 -1	6.5313 -1	1.8967 -2	
9800	1.6827 -1	6.7298 -1	1.7258 -2	
9900	2.0045 -1	6.9343 -1	1.5776 -2	
6000	2.3714 -1	7.1205 -1	1.4421 -2	

TABLE 6 REDUCED ABSOLUTE FORMATION ENTHALPIES  $\begin{bmatrix} \Delta^{H}_{f,abs} \\ \overline{\mathcal{R}} \end{bmatrix}^{T}_{i} = \begin{bmatrix} \Delta^{E}_{o,f} \\ \overline{\mathcal{R}} \end{bmatrix}^{T \circ K}_{i} \cdot \frac{1}{T} + \begin{bmatrix} H - E_{o} \\ \overline{T} \end{bmatrix}^{T}_{i}$ 

(x)		0	OII	MgQ
100	202.3700	299.5070 151.1550	50.3990	-a43.409
298.16	89.6411	102.87%	27.1010	-139.74/C
400	67.4550	76.8995	19.2123	-21.9917
500	54.4640	(8.0330	15.3775	-91.991 -67.824 -93.485
600 700	45.8033 39.6171	\$2.1190 45.0351	11.4227	-43.8033
800	34.9775	39.7273	9.4598	-31.4570
900	31.3689 24.4620	35.5893	8.8104	-27.67%
1100	26.1200	29.5754	7.8800	-21.740
300	24.1517 22.4862	27.3205 25.4116	7.5368	-19.49F
1500	21.0586 19.8213	23.7756 22.3580	7.0113	-15.002 -14.506
1600	18.7388	ยามก	6.6299	-13.2460
1700	17.7835 16.9344	20.0228	6.4770	-12.171
1900	16.1747	18.1707	6.3622 6.2255 6.1225	-10.217
100	14.8724	16.6857	6.0302	-8.558
200	14.3100	16.0412	5.9480 5.8736 5.8069	-7.98N
1400	13.3258	14.9128 14.4170	5.8069	-7.340 -6.70
1500			5.7470	-6.250
1600 1700	12.4931	13.9593	5.6920	-5.763
800	11.7793 11.4593	13.5354	5.5976	-5.3100 -4.886
900	11.1607	12.7752	5.5561	-4.115
100	10.8813	12.1128	5.4821	-3.7667 -3.4380
300 400	10.3733	11.5325	5.4197	-3.1279
NOO 1500	10.1418 9.9234	11.2669	5.3980 5.3667	-2.835 -2.5593
600	9.7172	10.7818	5.3486	-2.297
700 1800	9.5222 9.3374	10.5560	5.3195	-1.313
900	9.1621 8.9955	9.9593	5.2797	-1.5898 -1.3750
100	8.8371 8.6862	9.7782	5.2442	-1.1716
300	8.5423	9.6069	5.2261	-0.97%
100 500	8.4050	9.2876	5.1995 5.1859	-0.790 -0.619 -0.442
600	8.1483	8.9963 8.8610	5.1733	-0.2800
700	8.0281 7.4029	8.8619 8.7304	5.1617 5.1900	-0.1930 0.0973
900	7.9129 7.8024 7.6964	8.6052	5.1401	0.172
100			5.1300	0.3100
200	7.5945 7.49% 7.4023	8.2/05 8.2/07 8.1539	5.1207	0.57%
900 400	7.3115 7.9840	6.0;22	5.1031	0.445
900		7.9543	5.0010	0.9343
Kan yan Huu	7.1396	7.77.77	5.07th	1.1946
1900 1900	6.9191	7.0101 7.55h	5.0700	1.299
000	6.6303	7.5135	5.050	1.454
	WO THE			

# TABLE 7 REDUCED ABSOLUTE FORMATION ENTHALPIES $\begin{bmatrix} \Delta H_{f,abs} \\ \overline{R} T \end{bmatrix}_{i}^{T} = \begin{bmatrix} \Delta E_{o,f} \\ \overline{R} \end{bmatrix}_{i}^{T oK} \cdot \frac{1}{T} + \begin{bmatrix} H - E_{o} \\ \overline{T} \end{bmatrix}_{i}^{T}$

fig.et		the same of the sa	-	
100	111.7440	572.6300	-133.3930	-460.369
298.16	57.7490 39.9989	195.3/46	-142.402	-154.8137
300	38.7037	101.5100	-42.1291	-153.8690
500	30.6765 25.2670	144.0325	-30.7190	-114.1 <b>5%</b> -90.340
600	21.6743	96.8550	-19.2833	-71. 3040
700	19.1211	83.3757	-15.9993	-(2.830)
900	17.2178	73.2663 65.4033	-13.5250 -11.5919	-51.269 -117.941
1000	14.5780	59.1130	-10.0370	-42.1-24
1100	13.6273 12.6392	53.9664 19.67/5	-8.7586 -7.6867	-34.062
1300	12.1759	46.0485	-6.7192	-30.86
1400	11.6106	42.9379	-5.9961	-26.165
1500	11.1223	40.2420	-5.3153	-25.629
1700	10.6974	37.8831 35.8018	-4.7170 -4.1878	-23.769
1800	9.9924	33.9017	-3.7164	-20.3
1900	9.6967	32.2963	-3.2922	-18.8:0
2000	9.4315	30.8065	-2.9110	-17.5to
2100	9.1934 8.9767	29.4586 28.2332	-2.5641 -2.2478	-16.357
2300	8.7792	27.1143	-1.9593	-15.319
2400	8.5986 8.4336	26.0888	-1.6933	-13.430
2500		25.1452	-1.4492	-12.593
2700	8.2815 8.1406	24.2752	-1.7226	-31.829
2800	8.0098	22.7199	-0.8176	-10.4:29
2900 3000	7.6688	22.0237	-0.6350 -0.4647	-9.8:5
3200	7.6705	20.7653	-0.3055	-8.717
3500	7.5727	20.1956	-0.1555	-8.213
3300 3400	7.4794	19.6605	-0.0149	-7.73
3500	7.3922	19.2109	0.1161	-7.26 A
3600	7.2337	18.2348	0.3618	-6.4CM
3700 3800	7.1616	17.8118	0.4745	-6.0.2
3900	7.0270	17.0312	0.6823	-5.72. -5.360
4000	6.9658	16.6713	0.7780	-5.0%
4200	6.9069	16.3280	0.8705	-4.747
4300	6.7984	15.6928	1.0407	-4.172
4400 4500	6.6988	15.3966 15.1147	1.1211	-3.900
4600	6.6526	14.8452	1.2703	-3.400
4700	6.6087	14.5883	1.3407	-3.2cm
4800	6.5668	14.3414	1.4083	-2.939
5000	6.4876	13.8796	1.5344	-2.513
5100	6.4505	13.6636	1.5941	-2.312
5300	6.4157	13.4551	1.6517	-2.120
5300 5400	6.3725	13.0649	1.7600	-3.7%
5500	6.3165	12.0003	1.8123	-1.5%
5600 5700	6.2%6	12.7035	1.0017	-1.415
5800	6,2789	12.3:19	1.970	-1.09
5900	6.2013	12 .00% 12 .00%	9.000 9.0457	-0.70
(APa, r)	10199	26613	-13660	-41006

TABLE 8 REDUCED SENSIBLE ENTHALPIES,

1	H - Eo	T
	RT.	i

(K)	H <sub>2</sub>		e,	0	OIL	H,O
100 200 298.16 300 400	3.420 3.427 3.41A 3.417 3.436 3.452	2.500 2.500 2.500 2.500 2.500 2.500	3.497 3.508 3.502 3.521	2.657 2.730 2.715 2.714 2.667	3.648 3.72 3.70 3.699 3.670 3.647	3.955 3.903 3.997 3.996 4.016
600 700 800 900	3.4% 3.4% 3.4% 3.4% 3.495 3.507	2.500 2.500 2.500 2.500 2.500 2.500	3.553 3.594 3.640 3.687 3.732 3.775	2.644 2.628 2.640 2.600 2.507	3.631 3.621 3.66 3.66 3.621	4.090 4.139 4.199 4.253 4.316
1100 1200 1300 1400 1500	3.520 3.534 3.550 3.568 3.567	2.500 2.500 2.500 2.500 2.500	3.816 3.853 3.859 3.972	2.589 2.583 2.577 2.572 2.568	3.630 3.661 3.666 3.666 3.669	4.361 4.563 4.563
1600 1700 1800 1900 2000	3.608 3.628 3.670 3.671 3.692	2.500 2.500 2.500 2.500 2.500	3.981 4.008 4.034 4.056 4.058	2.504 2.501 2.558 2.555 2.555	3.708 3.777 3.746 3.765 3.765	4.717 4.782 4.645 4.907
2100 2200 2300 2400 2500	3.714 3.736 3.757 3.778 3.799	2.500 2.500 2.500 2.500 2.500	4.105 4.127 4.148 4.168 4.188	2.550 2.548 2.546 2.544 2.543	3.804 3.823 3.841 3.859 3.877	5.085 5.081 5.135 5.187 5.28
2600 2700 2800 2900 3000	3.819 3.839 3.859 3.878 3.897	2.500 2.500 2.500 2.500 2.500	4.208 4.227 4.245 4.263 4.261	2.542 2.541 2.540 2.539 2.538	3.894 3.911 3.928 3.944 3.959	5.206 5.333 5.370 5.421 5.463
3100 3200 3300 3400 3500	3.934 3.934 3.952 3.969 3.907	2.500 2.500 2.500 2.500 2.500	4.298 4.315 4.332 4.348 4.364	2.537 2.537 2.537 2.536 2.536	3.974 3.989 4.003 4.017 4.031	5.503 5.542 5.580 5.616 5.651
3600 3700 90 3900 6000	4.003 4.020 4.037 4.053 4.069	2.500 2.500 2.500 2.500 2.500	h.379 h.395 h.409 h.424 h.436	2.536 2.537 2.537 2.537 2.538	4.044 4.056 4.069 4.081 4.093	5.685 5.717 5.749 5.779 5.809
\$200 \$300 \$400 \$400	4.084 4.100 4.115 4.130 4.145	2.500 2.500 2.500 2.500 2.500	4.452 4.465 4.479 4.491 4.504	2.539 2.539 2.540 2.541 2.542	4.104 4.115 4.126 4.137 4.147	5.837 5.865 5.892 5.943
4600 4700 4800 4900 9000	\$.160 \$.17\$ \$.168 \$.202 \$.216	2.500 2.500 2.500 2.500 2.500	4.516 4.598 4.540 4.551 4.568	2.543 2.545 2.546 2.547 2.549	4.157 4.167 4.176 4.186 4.186	5.967 5.991 6.014 6.037 6.058
5100 5800 5800 5800 5800	4.230 4.243 4.277 4.270 4.263	2.500 2.500 2.500 2.500 2.500	4.573 4.584 4.594 4.604 4.614	2.550 2.559 2.553 2.555 2.557	h.20h h.213 h.221 h.230 h.236	6.000 6.100 6.140 6.140
5/00 5/00 5/00 5/00 600	h.296 h.309 h.321 h.3.96	2.500 2.500 2.500 2.500 2.500	4.633 4.633 4.631 4.631	2.5% 2.5% 2.5%	h.29h h.29h h.29a h.270	6.212

 $\frac{\left(\frac{1}{2} + \frac{1}{2} \right)_{i}^{T} \cdot \left(\frac{1}{2} + \frac{1}{2} \right)_{i}^{T_{i}} + \frac{T - T_{i}}{100} \left[ \left(\frac{1}{2} + \frac{1}{2} \right)_{i}^{T_{i}} - \left(\frac{1}{2} + \frac{1}{2} \right)_{i}^{T_{i}} \right] }{T_{i} < T < T_{i} + 100 \cdot T_{i}}$ 

TABLE 9 REDUCED SENSIBLE ENTHALPIES,

T H - E	7 1
H - E	- i

(K)	H <sub>7</sub>	10		00	en.
100	3.487	3.7%	2.501 2.499 2.501 2.499 2.500 2.500	3.467	3.49
200	3.495	3.754	2.499	3 405	3.57
200.10	3.499	3.710	2.501	3.449	3.77
400	3.500	3.679	2.500	3.501	3.77
200 298.16 300 400 500	3.495 3.499 3.497 3.500 3.508	3.707 3.679 3.669		3.497 3.501 3.512	4.25
600 700 800 900	3.520 3.541 3.565 3.594 3.624	3.476	2.500 2.500 2.500 2.500	3.530	4.46
800	3.541	3.64	2.500	3.555	4.04
900	3.594	3.719	2,500	3.555 3.505 3.617	4.99
1000	3.624	3.779		3.651	4.66
1200	3.655 3.606	3.810	2.500 2.500 2.500 2.500 2.500	3.685	5.27
1200	3.606	3.840	2.500	3.718	5.397 5.507 5.607
1300	3.746	3.847	2,500	3.781	5.60
1500	3.717 3.746 3.775	3.869		3.750 3.761 3.810	5.69
1600	3.802 3.828 3.852	3.948	2.500 2.500 2.500 2.500	3.83b 3.864 3.888	5.784
1700 1800	3.020 3.869	3.9/1	2.500	3.004	5.86
1900	3.875	3.993 4.013	2.500	3.912	6.001
2000	3.875 3.898	4.032		3.933	6.063
2100	3.918 3.938 3.957	4.051 4.068 4.084 4.099 4.114	2.500 2.500 2.500 2.500 2.500	3.954	6.120
5500	3.935	4.008	2.500	3.974	6.17
2300	3.975	4.099	2.500	3.974 3.992 4.010	6.27
2500	3.975 3.992	4.116		4.026	6.316
2600 2700 2800 2900 3000	4.008	4.128	2.501 2.501 2.502 2.502	4.042	5.356 6.396 6.435 6.470
2800	4.024	4.141	2.501	4.057	6.190
2900	4.038 4.052 4.066	4.153	2.500	4.085	6.470
3000		4.176		4.098	
3700	4.078 4.090 4.102	4.187	2.503 2.504 2.505 2.506 2.508	4.110	6.536 6.566 6.59 6.62
3200 3300 3400 3500	4.302	4.197	2,505	4.133	6.50
3400	4.113	4.216	2.506	4.133	6.62
		4.226	2.508	4.154	6.049
3600	4.134 4.134 4.163 4.172	4.234	2.509	4.164	6.67
3000	4.154	4,251	2.513	4.174	6.722
3700 3000 3900 4000	4.163	4.251 4.258 4.266	2.513 2.515 2.518	4 100	6.74
				4.200	
4200 4200	4.180 4.189	4.273 4.280 4.287	2.520 2.523 2.527 2.530 2.534	4.209	6.700
4300 4400 4500	4.197	4.287	2.527	4.224	6.80
N100	4.197 4.204 4.212	4.293	2.530	4.232	6.84
				4.239	6.00
4600 4700	4.219 4.226	4.305 4.311 4.317 4.323 4.328	2.538 2.543 2.547 2.552 2.557	4.246	6.879
1700 1800 1900 5000	4.233 4.240 4.246	4.337	2.547	4.253	6.912 6.928 6.944
4900	4.240	4.323	2.552	4.266	6.92
				4.272	6.944
5100 5200 5300 5400 5500	4.253 4.259 4.265 4.271	4.333 4.339 4.335 4.348 4.353	2.563 2.568 2.574 2.581 2.587	4.278	6.955
5300	4.265	4.335	2.574	4.200	6.44
5400	4.272	4.348	2.581	4.301	7.00
	4.270				7.015
5600	4.262 4.205 4.293	4.358 4.371 4.371 4.375	2.594 2.600 2.607	4.306 4.311 4.317	7.012
5700 5700 5900 6000	4,293	4.30	2,607	4,317	7.05
5900	4.298	4.371	2.614	4.327	7.0
(000	4.303	4.375	2.622	4.327	7.075

 $\frac{\left(\frac{H-R_0}{4T}\right)_1^{T_1} \cdot \left(\frac{H-R_0}{4T}\right)_1^{T_1} + \frac{T-T_1}{100} \left[ \left(\frac{H-R_0}{4T}\right)_1^{T_2} \cdot \left(\frac{H-R_0}{4T}\right)_1^{T_2} \right] }{T_1 < T < T_1 + 100 + T_2}$ 

TABLE 10 REDUCED ENTROPTES ,

1	s <sup>p=1</sup>	TT
1-	R	i

(K)	No	11	e.		CH	1141
100	12.2/2	11.053	20.894	16.357	14.041	18.31
200	14.351	12.73	23.205	3H.; Hc	20.648	21.00
2-11.26	15.704	13.70%	24.659	19.357	25.300	55 .(4)
300	15.720	13.799	24.641	19.374	22.122	22.72
400	16.730	14.518	25.709	20.184	23.150	23.84
500	17.515	15.07/	26.530	20.097	23.44	24.8
600	18.157	15.53	27.222	23 .162	24.590	25.60
700	18.702	15.917	27.826	21.553	25.149	26.24
	19.170	10.272	211.3.11	21.80	25.016	54.70
900	19.598	16.609	29.283	22.167	21.426	27.91
1100	20.326	17.048	29.485	22.692	26.752	28.45
1200	20.326	17.265	30.056	22.910	27.110	24.40
	20.947	17,465	30.401	23.311	27.416	20.33
1300	20.947	17.00	30.723	23.297	27.704	29.33 29.73
1500	21.495	17.103	30.723	23.471	27.915	30,12
1600	21.747	17.984	31.310	23.632	28.232	30.49
1700	21.988	18.136	31.580	23.70	28.476	30,84
1800	22.217	18.279	31.635	83.907	28.700	31.18
1900	22.436	18.414	32.079	24.063	28.932	31.50
2000	22.646	18.542	32.311	24.191	29.145	37.65
2100	22.849	18.664	32.533	24.313	29.349	32.12
2200	23.043	18.781	32.747	24.430	29.545	12.41
2300	23.231	18.891	32.951	24.541	29.734	32.69
2400	23.413	18.998	33.149	845.45	29.916	32.9
2500	23.588	19.100	33.339	24.760	30.092	33.22
2600	23.758	19.198	33.524	24.849	30.201	33.46
2700 2800	23.922	19.292	33.702	24.943	30.425	33.73
2900	24.002	19.383	33.074	25.035	30.584	33.97
3000	24.237	19.555	33.874 34.042 34.204	25.209	30.738	33.9/ 34.20 34.43
3100	24.534	19.638	24 262	25.291	31.033	
3200	24.534	19.717	24 516	25.371	31.174	34.65
3300	24.817	19.79	34.665	25.371	91 911	35.07
3300	24.952	19.869	34.811	25.529	31.444	35.27
3500	24.952 25.085	19.941	34.953	25.525	31.575	35.47
3600 3700 3800 3900 4000	25.214	20.011	35.092	25.670	31.701	35.66
3700	25.341	20.080	35.227	25.739	31.825	35.85
3800	25.465	20.147	35.360	25.807	31.945	35.85 36.0h
3900	25.341 25.465 25.586 25.705	20.212	35.360 35.489	25.873	31.945	36.22
4000	25.705	20.275	35.615	25.938	32.178	36.39
1100	25.821	20.337	35.739	26.002	32.291	36.50 36.73 36.90
4200	25.935 26.047	20.397	35.859 35.978	56.0CH	32.401	36.73
4300 4400	26.017	20.456	35.978	26.124	32.509	36.90
4500	26.157 26.264	20.513	36.004	26.242	32.717	37.06
4600						
4700	26.370	20.624	36.319 36.429 36.536 36.640	26.299 26.354 26.409 26.463	32.819	37.37
4700 4800	26.577	20.070	¥. 426	26.400	33.015	37.53
4900	26.577	20.731	16.640	26.463	33.111	37.62
4900 5000	26.716	80.833	36.744	26.516	33.805	37.97
5100	25.874	20.883		26.56B	33.291	30.11
5200	26.969	20.931	36.845	26.568 26.619	33.3/37	20.2
5300	26.969 27.064	20.919	37.042	86.669	33.476	W. W
5300 5400	87.156	21.025	37.138	26.719	33.563	34.50
5500	27.156	\$1.0/1	37.233	26.768	33.49	38.60
5600 5700 5800	27.427	21.116	37.325	26.815	33.733	38.7
5700	27.427	21.100	37.416	20,.103	33.733	38.91
5840	27.515	21 .POA	37.596 37.596 37.682	96.919	33.1194	39.03
5900	27.601	71.247	37.5%	25.454	33.5/19	33.30
FAM	27.CB6	21.200	W/ ('BO	26.990	N.057	32.8

TABLE 11 REDUCED ENTROPIES,

(K)	N <sub>2</sub>	NO		co	co <sub>2</sub>
100		21.579	15.693	19.885	21.77
500	19.179	23.091	17.437	22.345	24.05
298.16	23.039	25.335	17.427 18.424 18.440	23.758	25.70
298.16 300 400	23.053	25.359	18.440	23.780	25.72
400	23.053 24.062 24.851	25.359	19.159	24.790	27.08
500		27.200	19.717	22.355 23.758 23.780 24.790 25.402	28.24
600 700 800	25.505 26.069 26.568	27.876 28.462	20.173 20.558 20.892 21.186	26.243	30.14
800	26.568	28.983	20.802	27. 221	30.95
900	27.018	29.452	21.186	27.321 27.777	31.69
900	27.428	29.879	21.450	28.194	32.37
1100	27.806 28.157 28.483	30.271	21.688	28.577	33.00
1200	28.157	30.63 <sup>1</sup> 30.971 31.287	21.906	28.932	33.59 34.13 34.65
1300	28.403	30.971	55.100	29.263	34.13
1300 1400 1500	28.789	31.583	22.891 22.464	29.573	35.13
1600	29.349	31.861	22.625	30.138	35.58
1700	29.606	32,125	22.777	30.398 30.644	36.01 36.42
1800	29.020	32.374 32.611	22.919	30.644	36.42
1900	30.083 30.304	32.611	22.919 23.054 23.183	30.878	36.81
2100		32.851		31.314	37.54
2200	30.515	33.258	23.305	31.518	37.54 37.88
2300	30.912	33.455	23.532 23.639	31.714	35.20
2400	31.099	33.645 33.827	23.639	31.902	38.52
2500	31.279		23.741	32.082	
2600	31.619	34.002 34.172 34.335 34.492	23.839 23.934 24.026	32.256	39.11
2700	31.780	34.276	2h.096	32.586	39.39
2900	31.9%	34.400	24.114	\$ .743	39.00
3000	31.936 32.087	34.645	24.199	32.586 32.743 32.895	39.39 39.92 40.18
3100	32.233	34.793 34.936	24.202	33.041	40.42
3200	32.375	34.936	24.362	33.184	40.66
3500	32.513 32.646	35.075	24.440	33.322	40.89
3300 3400 3500	32.776	35.342	24.590	33.587	41.33
3600	32.903	35.470	24.663	33.724	41.55
3700	32.903 33.006	35.594 35.716	24.733	33.714	41.55
3800	33.147	35.716		33.959 34.076	41.90
3700 3800 3900 4000	33.264 33.378	35.834	24.869	34.076	42.15
<b>\$100</b>			25.001		42.53
M200	33.490 33.599 33.706	36.062 36.171	25.065	34.303 34.413 34.519	42.72
4300	33.706	36.279	25.127	34.519	42.90
4300 4400 4500	33.800	36.384 36.487	25.189	34.624	43.07
	33.912		25.250	31.727	43.25
4600	34.012	36.588 36.686	25.310	34.827	48.45
M800	34.206	36.783	25.369 25.427	35.021	43.56
4900	34.299	30.077	25.484	35.116	43.90
4700 4800 4900 5000	34.392	36.971	25.541	35.208	44.05
5100	34.482 34.572 3 658	37.002	25.597 25.652	35.299	44.21
5200	34.572	37.151	25.652	35.388	144.30
5300 5400	3 658	37.238	25.707	35.475	14.50
5500	3 658 34.744 34.828	37.40)	25.762	35.561	44.69
9600	34.911	37.402	25.869	35.728	44.93
5700	34.992 35.072 35.151	37.492	25.969 25.988	37.810	49.00
5800	35.072	37.694	25.9/4 26.026	35.690	45.20 45.20
5900 6000	35.151	37.733	20.026	35.769	45.36
	35.228	37.811	26.077	36.04G	45.47

TABLE 12 REDUCED SPECIFIC HEATS,

_	c 7	1 m
	Cp	1 -
1	C <sub>p</sub>	li

The second secon					
(K)	I <sub>e</sub>	ч	0-	0	OH
100 200 298.16 300 400	2.714	2.500	3.501	2.6-31	3.90%
500	3.200	2.500	3.503	2.735	
295.16	3.468	2.500	3.533	2.035	3.107
500	3.409	2.500	3.534	9.634	3.100
500	3.280 3.468 3.469 3.510 3.529	2.500 2.500 2.500 2.500 2.500 2.500	3.501 3.503 3.533 3.534 3.621 3.739	2.557	3.407 3.400 3.546 3.550
600	3.527			2.541	3.551
700	3.541	2.500	3.907	2.532	3.90
800	3.50	2.500	4.058	2.524	3.598
600 700 800 900	3.527 3.541 3.56 3.597 3.633	2.500 2.500 2.500 2.500	3.860 3.967 4.058 4.133 4.195	2.531 2.524 2.519 2.516	3.551 3.566 3.660 3.690
1100 1200 1300 1400	2 674	2.500 2.500 2.500 2.500	h.267	2.513	
1200	3.719 3.769 3.825 3.805	2.500	4.291 4.330 4.365 4.397	2.511 2.510 2.508 2.507	3.744 3.799 3.854 3.908 3.959
1300	3.769	2.500	4.330	2.510	3.854
1400	3.025	8.500	4.365	2.508	3.908
		(4500)			
1600 1700 1800 1900 2000	3.937 3.906 4.034 4.080 4.184	2,500 2,500 2,500 2,500 2,500	4.428 4.458 4.467 4.515 4.544	2.507 2.506 2.506 2.505 2.505	4.007
1800	4.034	2.500	4.487	2.506	4.095
1900	4.080	2.500	4.515	2.505	4.053 4.095 4.134 4.170
					4.170
2100	4.166 4.206 4.244 4.280 4.315	2.500	4.571 4.599 4.627 4.654 4.681	2.506 2.506 2.506 2.507 2.508	4.203
2300	4.244	2,500	4.627	2,506	4.263
2300 2300 2500	4.280	2.500 2.500 2.500 2.500	4.654	2.507	4.235 4.263 4.291 4.339
2600 2700 2800 2900 3000	4.347 4.378 4.407 4.433 4.438	2,500 2,500 2,500 2,500	4.707 4.733 4.758 4.762 4.806	2.509 2.511 2.513 2.516 2.518	4.350 4.360 4.400 4.418
2800	1.570	2.500	4.733	2.511	4.360
2900	4.433	2.500	4.782	2.516	4,400
					4.418
3100 3200 3400 3500	4.510 4.535 4.560 4.584	2.500 2.500 2.500 2.500	4.829 4.851 4.872 4.893 4.913	2.521 2.525 2.529 2.533 2.537	4.435
3300	4.500	2.500	4.872	2.00	1.452
3400	4.960	2.500	4.693	8.533	4.481
				2.537	4.452 4.467 4.481 4.495
3600 3700 3800 3900	4.609 4.632 4.656 4.679 4.701	2,500 2,500 2,500 2,500 2,500	4.931 4.949 4.966 4.988 4.998	2.541 2.546 2.551 2.557 2.562	4.508 4.581 4.533 4.545 4.556
3/00	4.632	2.500	4.949	2.546	4.521
3900	4.679	2.500	4.962	2.557	4.545
	4.701				
4200 4200 4300 4400 4500	4.723 4.745 4.767 4.768 4.808	2.500 2.500 2.500 2.500 2.500	5.013 5.026 5.040 5.052 5.063	2.568 2.574 2.580 2.586 2.592	4.566 4.577 4.507 4.506
4200	1.745	2.500	5.026	2.574	577
N400	4.700	2,500	5.050	2.986	1.507
4600 4700 4800 4900 9000	4.848 4.848 4.868 4.867 4.905	2.500 2.500 2.500 2.500	5.075 5.005 5.094 5.103 5.111	2.598 2.604 2.610 2.616 2.622	4.635 4.633 4.641 4.649
A700	4.045	2.500	5.085	2.604	4.624
1900	4.887	2.500	5.103	2.616	1.641
	4.905	2.500	5.111		1.69
5100	943 943 951	2.500	5.119	2.628 2.634 2.646 2.646 2.632	4.657
7000	1.93	8.500	5.126	2.634	1.63
2300	4.701	2.500	5.133	2.00	1.673
5100 9800 5300 5400 5500	4.997	2.500 2.500 2.500 2.500 2.500	5.119 5.126 5.133 5.139 5.146	2.652	4.657 4.673 4.601 4.601
9600 9700 9800 9900	5.005	2.500 2.500 2.500 2.500 2.500	5.150 5.157 5.160 5.167 5.171	9.663 9.663 9.668 9.674 9.679	4.605 4.708 4.709 4.716
5700	5.038 5.049 5.066 5.083	2.500	5.157	2.663	1.70
9000	3.009	2.500	5.162	9.608	4.709
3700	7.000	9.600	7.107	2.079	6.723

 $\left(\frac{2}{3}\right)_{2}^{2} \cdot \left(\frac{2}{3}\right)_{3}^{2} \cdot \frac{1}{1000} \left[\left(\frac{2}{3}\right)_{10}^{2} \cdot \left(\frac{2}{3}\right)_{3}^{2}\right]$ 

TABLE 13 REDUCED SPECIFIC HEATS,

[con	T		
मि	i		

1.				-		
(K)	<b>N</b> .	-	W.	CO <sup>5</sup>	co	H-O
500 700	3.500	2,500	3.865	3.513 3.692 4.466	3.500	4.00
598.16	3.503	2,500	3.500	4.466	3.405	4.03
300	3.503		3.500	9.477	3.505	4.03
900 900 898.16	3.503 3.503 3.518 3.557	2,500	3.601	5.307	3.505 3.529 3.533	4.23
600	. /		3.757	5.692 5.961 6.186	3.60	4.36 4.50 4.65 4.80
700	3.699 3.760 3.660	2.500	3.852	5.961	3.749	4.50
900	3.860	2.500	3.947	6.374	3.918	4.80
1000		2.500 2.500 2.500 2.500	3.052 3.941 4.020 4.068	6.532	3.991	4.95
1200	3.998	2.500 2.500 2.500 2.500 2.500	4.146	6.664	4.055	5.10
1300	4.107	2.500	4.237	6.072	4.158	5.41
1300	4.153	2.500	4.195 4.237 4.273	6.952	4.200	5.52
1500			4.304	7.022	4.230	5.65
1600	4.226	2.500	4.353	7.032	4.257	5.76
1700	4.256	2.500	4.374	7.134	4.295 4.319 4.341	5.97
1900 R000	4.307	2.501	4.330 4.353 4.374 4.392 4.408	7.222	4.360	5.97 6.06 6.14
2100	4.364 4.360 4.394 4.406	2.501	4.422	7.291		6.22
2200	4.354	9 600	4.435	7 920	4.377 4.392 4.406	6.29
2300 2400	4.300	2.502	4.458	7.347	4.406	6.41
2500	4.406	2.505	4.467	7.393	4.430	6.47
2600	4.418	2.507	4.476	7.415	4.441	6.52
2700 2800	4.438	2.510	4.485	7.451	4.451	6.61
2900 3000	4.438 4.456	2.517	4.506	7.433 7.451 7.468 7.485	4.468	6.69
3300		9 697	4.513		4.484	
3200 3300 3400	4.464	2.534	4.519	7.513	4.401	6.73
3300	4.479	2.541	4.525	7.526	4.497	6.79
3500	1.492	2.550	4.525 4.530 4.535	7.499 7.513 7.526 7.539 7.551	4.503	6.85
3600	4.498 4.504 4.510	2.570 2.582 2.595 2.608	4.540 4.545 4.550 4.554 4.558	7.564 7.575 7.586 7.597 7.608	4.515	6.87
3700 3800 3900	4.510	2.502	4.545	7.575	4.521	6.90
3900	4.515	2.608	4.554	7.597	4.531	6.90 6.92 6.94 6.97
	4.521	2.623				6.97
4100 4200	4.526	2.639	4.562 4.566 4.570	7.618	4.541	7.00
4300	4.535	2.656	4.570	7 639	4.550	7.00
4300 4400 4500	4.535 4.540 4.544	2.693	4.574	7 639 7.648 7.657	4.555	7.06
hem		2.733			4.563	7.07
1700 1800 1900 5000	4.549 4.553 4.558 4.562 4.566	2.733 2.754 2.776	4.582 4.585 4.589 4.592 4.595	7.666 7.676 7.685 7.694 7.702	4.563 4.567 4.571	7.09
4900	4.562	2.799	4.500	7.694	4.575	7.10
5000	4.566	2.799	4.595	7.702	4.579	7.13
5100 5200	4.571	2.846 2.870 2.894	4.599	7.713	4.583	7.14
5300	4.579	2.894	4.602	7.735	4.591	7.17
5300 5400 5500	4.579 4.584 4.588	2.919	4.602	7.735 7.746 7.757	4.591	7.18
5600			4,615		4.602	7.21
5700 5800	4.593 4.598 4.600	2.969 2.974 3.019 3.045	4.61B	7.768	4.606	7.22
5000	4.607	3.019	4.624	7.790	4,610	7.23
51,00	4.612	3.070	4.627	7.813	4.617	7.25

$$\frac{\left(\frac{C_{L}}{q}\right)_{1}^{T} \cdot \left(\frac{C_{L}}{q}\right)_{1}^{T_{1}}}{T_{1}} \cdot \frac{T_{1} - T_{1}}{1(T_{1})} \left[\left(\frac{C_{L}}{q}\right)_{1}^{T_{R}} \cdot \left(\frac{C_{L}}{q}\right)_{1}^{T_{1}}\right]$$

$$T_{1} < T < T_{1} + 100 \text{ s. } T_{2}$$

	(K)	H <sub>2</sub>	н	٥,	•	ОН	N.P.
	100 200 298.16 300 400 500	2.0018 2.7080 2.752 2.7571 2.7923 2.8186	2.4001 2.4130 2.4193 2.4193 2.4231 2.4231	4.9371 4.3910 4.3779 4.3271 4.2909 4.2000	3.5475 3.4513 3.3974 3.3957 3.3588 3.358	3.8783 3.8783 3.8785 3.8785 3.8735 3.8739	3.9771 3.98cy 3.9838 3.9838 3.9860 3.9946
	600 700 800 900 1000	2.8304 2.8548 7.8687 7.8510 2.8521	2.4200 2.4297 2.4313 2.4324 2.4334	4.2555 4.2475 4.2403 4.2391	3.30ft 3.2900 3.2747 3.2616 3.2704	3.8440 3.63/2 3.8321 3.1284 3.82:8	6.0033 6.0136 6.0366 6.0368 6.0697
50 50 50 50	1100 1200 1300 1400 1500	2.9024 2.9122 2.9214 2.9305 7.9392	2.4344 2.4351 2.4358 2.4364 2.4371	4.2389 4.2392 4.2400 4.2410 4.2425	3.8403 3.2313 3.2232 3.210 3.2094	3.8043 3.803 3.8043 3.8043 3.8053	4.0631 4.0702 4.0907 4.1040
	1600 1700 1800 1900 8000	2.9476 2.9560 2.9540 2.9718 2.9794	2.4376 2.4382 2.4386 2.4391 2.4394	4.2438 4.2455 4.2472 4.2491 4.2509	3.2031 3.1975 3.1922 3.1673 3.1886	3.8266 3.8283 3.6301 3.8323 3.8344	4.1327 4.1465 4.1599 4.1733 4.1863
	2100 2200 2300 2400 2500	2.9870 2.9941 3.0012 3.0081 3.0148	2.4398 2.4403 2.4405 2.4409 2.4412	4.2529 4.2550 4.2569 4.2590 4.2611	3.1783 3.1743 3.1704 3.1668 3.1633	3.8366 3.8389 3.8413 3.8437 3.8461	4.1991 4.2114 4.2237 4.2354 4.2470
	2600 2700 2800 2900 3000	3.0214 3.0277 3.0340 3.0401 3.0461	2.4415 2.4417 2.4420 2.4423 2.4426	4.2634 4.2655 4.2677 4.2699 4.2721	3.1601 3.1569 3.1541 3.1512 3.1486	3.8484 3.8508 3.8532 3.8555 3.8579	4.2583 4.2692 4.2796 4.2901 4.3003
	3500 3500 3500 3100	3.0518 3.0575 3.0632 3.0685 3.0739	2.4430 2.4432 2.4432 2.4435 2.4436	4.2743 4.2766 4.2767 4.2610 4.2632	3.1460 3.1435 3.1412 3.1390 3.1368	3.8625 3.8625 3.8648 3.8669 3.8692	4.3197 4.3197 4.3290 4.3361 4.3470
	3600 3700 3600 3600	3.0791 3.0843 3.0894 3.0943 3.0992	2.4440 2.4442 2.4444 2.4445	4.2854 4.2876 4.2898 4.2920 4.2940	3.1348 3.1328 3.1309 3.1290 3.1273	3.8713 3.8735 3.8755 3.8776 3.8796	4.3598 4.3741 4.3784 4.3805 4.3883
	#300 #300 #400	3.1040 3.1067 3.1133 3.1179 3.1223	2.445 2.445 2.445 2.445 2.445	4.2962 4.2962 4.3003 4.3024	3.1257 3.1241 3.1225 3.1210 3.1197	3.8617 3.8637 3.8657 3.8675 3.8694	4.4035 4.4109 4.4181 4.4292
	4600 4700 4600 4900 9000	3.1267 3.1312 3.1354 3.1396 3.1438	2.4456 2.4457 2.4458 2.4450	4.3064 4.3084 4.3103 4.3121 4.3241	3.1183 3.1169 3.1196 3.1144 3.1132	3.8932 3.8932 3.8949 3.6958 3.8986	h.4320 h.4367 h.6456 h.6518 h.4582
	\$100 \$800 \$300 \$400 \$900	3.1480 3.1519 3.1560 3.1590 3.1639	2.4462 2.4464 2.4464 2.4466 2.4466	4.3159 4.3177 4.3195 4.3813 4.3831	3.1121 3.1110 3.1099 3.1070 3.1000	3.9003 3.9000 3.9037 3.9070	h. h6h5 h. h706 h. h765 h. h623 h. h681
	9600 9700 9800 9900 6000	3.1676 3.1724 3.1722 3.1769 3.1805	2.4467 2.4467 2.4469 2.4470 2.4472	4.3°46 4.3°44 4.3°41 4.3891	3.1070 3.102 3.103 3.1043 3.103	3.90% 3.90% 3.900 3.900 3.900 3.900	4.4938 4.4993 4.5644 4.5108 4.5255

TABLE 15 REDUCED ENTROPY DIVIDED BY In T.

				<b> </b> ;	In T		
7.					1		
(x)	N <sub>2</sub>	NO			CO.		
300	4.16k7 4.0826	4.000	3.1077 3.2892	4.3180	4.540		
298.16	4.0424	4.4466	3.233	4.2698	4.510		
300	4.0417	4.4458	3.2329	4.1692	4.5105		
500	4.0160 3.9988	4.4016	3.197/	4.1376 4.1160	4.5211		
G00 700	3.9871 3.9793	4.3577 4.3446	3.1535	4.1024	4.6015		
800	3.9745	4.3358	3.1254	4.00/1	3.6312		
900	3.9718	4.3297	3.1145	4.0834	4.659		
1100	3.9705	4.3225	3.0969	4.0806	4.7132		
1300	3.9713	4.3207	3.0997	4.0806	4.761		
1400	3.9725	4,3189	3.0831	4.0023	4.7831		
1500	3.9760	4.3186	3.0717	4.000	4.8039		
1600	3.9780	4.3185	3.0666	4.0850	4.823		
1800	3.9802 3.9824 3.9847	4.3191	3.0577	4.0883	4.860		
1900 2000	3.9847 3.9869	4.3196	3.0537 3.0500	4.0900	4.876		
5700	3.9890	4.3207	3.0465	4.0935	4.907		
2200	3.9913	4.3213	3.0432	4.0953	4.9223		
2300	3.9935	4.3228	3.0372	4.0971	4.936		
2500	3.9978	4.3235	3.0344	4.1004	4.9621		
2600 2700	3.9997	4.3242	3.0317	4.1021	4.974		
2800	4.0038	4.3257	3.0269	4.1054	4.9971		
900 3000	4.0058	4.3263	3.0247	4.10/0	5.008		
3100	4.0095	4.3279	3.0205	4.1100	5.0286		
3200 3300	4.0131	4.3286	3.0185	4.1116	5.0383		
3400	4.0147	4.3302	3.0149	4.1145	5.0569		
3900	4.0164	4.3309	3.0133	4.1158	5.0657		
3600 3700	4.0181	4.3316	3.0118	4,1171	5.0826		
3800	4.0213	4.3330	3.0089	4.1199	5.0907		
3900 4000	4.0229	4.3337	3.00%	4.1211	5.0986		
1100	4.0258	4.3350	3.0054	4.1236	5.1136		
4200 4300	4.0273	4.3356	3.004	4.1249	5.1209		
1400	4.0301	4.3363	3.0033	4.1259	5.1279		
4500	4.0314	4.33/6	3.0017	4.1284	5.1416		
4600 4700	4.0328	4.3383	3.0010	4.1294	5.1484		
4800	4.0355	4.3395	2.9998	4,1316	5.1600		
1900 5000	4.0366	4.3400	2.9992	4.1328	5.1670		
2200	4.0391	4.3413	2.9984	4.1348	5.1789		
5200 5300	4.0404 4.0415		2.9984	4.1358	5.1903		
5400	4.0427	4.3430	2.9917	4.1378	5.1957		
5500	4.0439		2.9974	4.1378	5.2013		
5600 5700	4.0461	4.3447	2.9974	4.1407	5.206		
5800	4.0473	4.34.2	2.7774	4.3437	5.2167		
5900 6000	4.0484	4.3458 4.3463	2.99/5	4.1476	5.2271		
17 18.15	67	38 113		1 7:	13		
1 - 4	1/5	')]''	/[(aFe)]	LS [\abor)	1"		
(3 mi)	- 647	1 . 1	1/2	. 1	1. )}		
(7)	- 60.7	7			11		

TABLE 16

FLUID PROPERTIES OF SUPERSONIC COMBUSTION RAMJET SHOCKED INLET (CRITICAL CONDITION)

	FREE	FUSELAGE	DIFFUSER	DIFFUSER
	STREAM	AIRSTREAM	INLET	THROAT
		1	2	3
T (K)	200	230	1256.6	1110.5
P (atm)	.01	.02	. 59236	.3565
u (m/s)	1420.3	1525	281.315	652.274
M (frozen)	5	5	0.407	1.000
<b>n</b> <sub>02</sub>	.210084	.210084	.209922	.210032
7 N <sub>2</sub>	.789916	.789916	.789752	.789864
70	0	0	0	0
n N	0	0	0	0
no No	0	0	.000326	.000104
<b>7</b> H <sub>2</sub>	0	0	0	0
7 H20	0	0	0	0
<b>₹</b> OH	0	0	0	0
<b>₹</b> H	0	0	0	0
m(g/mole)	28.853	28.853	28.853	28.853
(frozen)	1.4	1.4	1.319167	1.32772
SR2	NC	NC	29.7868	29.7862
h <sub>f abs</sub>	NC	. 556455	3.742178	3.263834
<b>₹</b> 3	NA	NA SERVICE SERVICE	1.5782	398.

NC-means Not Calculated NA-means Not Applicable

TABLE 17

FLUID PROPERTIES OF SUPERSONIC COMBUSTION RAMJET SHOCKED INLET,  $M_1=5$ ,  $0_0^{\circ}=0.5$ 

	DIFFUSER	COMBUSTION	CHAMBER	EXHAUST
	EXIT	INLET	EXIT	NOZZLE EXIT
	3c	3c	4	5
8,6				
T (K)	739	739	2428	2123
P (atm)	.0722	.0722	.0722	.02
u (m/s)	1125.78	1125.78	1125.78	1813.105
M (frozen)	2.093	NC	1.088	1.896
702	.210084	.147929	.0129	.0071
<b>₹</b> N2	.789916	.556213	.627008	.642571
<b>₹</b> 0	0	0	.004220	.000964
₹ <sub>NO</sub>	0	o .	.004694	.001853
₹H <sub>2</sub>	0	.295858	.038280	.018524
<b>7</b> <sub>H20</sub>	0	0	.280390	.318257
noH 2	0	0	.019017	.0075696
$\eta_{\rm H}$	0	0	.013169	.003441
$\mathcal{M}(g/mole)$	28.853	NC	23.652848	24.203772
(frozen)	1.358717	NC	1.254343	1.253704
<u>s</u>	29.7868	NC	42.50791	42.5074
h <sub>f abs</sub>	2.101288	NC	2.876558	.087128
AA3	1.9028	1.9028	7.6261	14.6064

TABLE 18

FLUID PROPERTIES OF SUPERSONIC COMBUSTION RAMJET SHOCKED INLET, M =5.  $\partial_{0}^{\circ}$  =2

	DIFFUSER	COMBUSTION		EXHAUST
	EXIT 3c	INLET 3c	EXIT 4	NOZZLE EXIT
T (K)	967	967	1656.7	931
P(atm)	.205	.205	.205	. 02
u (m/s)	870.529	870.529	870.529	1623.397
M (frozen)	1.426	NC	1.085	2.717
n 02	.210084	.190114	.1486	.1497
7 N2	.789916	.714829	•749335	.750492
20	0	0	.000024	0
7 <sub>N</sub>	0	NC	NC	NC
no.	0	0	.002161	.000018
<b>7</b> H <sub>2</sub>	0	.095057	.000007	0
<b>7</b> <sub>H2</sub> 0	0	0	.099592	.099800
<b>₹</b> OH	0	0	.000379	o . ost
<b>P</b> <sub>H</sub>	0	0	.0000006	0 40
n(g/mole)	28.853	NC	27.614528	27.614592
(frozen)	1.3373	NC .	1.291421	1.330458
SR <sub>2</sub>	29.7840	NC	33.5544	33.5550
hf abs	2.804904	NC	3.065934	.473109
A A	1.1340	1.1340	2.0300	6.2703

FLUID PROPERTIES OF SUPERSONIC COMBUSTION RAMJET
SHOCKED INLET (CRITICAL CONDITION)

· 61 83849

	FREE	FUSELAGE	DIFFUSER	DIFFUSER
	STREAM	AIRSTREAM	INLET	THROAT
	0	1	2	3
T (K)	200	300	3084	2835
P (atm)	.01	. 04	3.162	1.046
u (m/s)	2840.5	2785	368.434	1014.94
M (frozen)	10	, <b>8</b> 41100	0.341	0.986
702	.210084	.210084	.16568	.17679
7 N2	.789916	.789916	.752654	.761866
70	0	0	.034057	.024984
₹ <sub>N</sub>	0 18130	0	.000011	.000004
₹ <sub>NO</sub>	0 70000	o vacae	.047598	.036356
<b>7</b> H2 00800	0 55555	0	0	0
₹ <sub>H2</sub> 0	0 97500	0	0	0
<b>₹</b> OH	0 300000	0	0	0
RH COLATA	0 ssept.	0	0 828.	0 (alak
n(g/mole)	28.853	28.853	28.36149	28.490545
Y (frozen)	1.4	1.4	1.289101	1.249
SR <sub>2</sub>	NC	NC	32.6595	32.6564
h <sub>f abs</sub> R <sub>2</sub> T <sub>2</sub>	NC	.340277	4.55986	4.065243
A A	NA	NA	. 9958	1

TABLE 20

FLUID PROPERTIES OF SUPERSONIC COMBUSTION RAMJET SHOCKED INLET, M =10,  $\mathbf{v}_{0_2}^{\circ}$  =0.5

	DIFFUSER	COMBUSTION	CHAMBER	EXHAUST
	EXIT	INLET	EXIT	NOZZLE EXIT
	3c	3c	.17.4	5
T (K)	2490	2490	3026.5	2223
P (atm)	.46	.46	.46	. 04
u (m/s)	1514.986	1514.986	1514.986	31 34. 385
M (frozen)	1.571	NC	1.247	3.198
n 02	.193418	.147929	.0222	.0082
n <sub>N2</sub>	.775113	.556213	.570000	.639646
70	.008879	0	.027032	.001414
₹ <sub>N</sub>	.000004	0	NC .	NC
No.	.022589	0	.014186	.002508
7 H2	0	.295858	.075977	.022755
7H20	0	0	.171442	.310772
₹ OH	0	0	.049762	.009909
₹ <sub>H</sub>	0	0	.069330	.004846
m(g/mole)	28.72488	NC	21.695834	24.09626
Y (frozen)	1.290383	NC	1.273074	1.252743
SR <sub>2</sub>	32.6600	NC	41.36155	41.38221
hf abs	3.365602	NC	4.500086	0.336135
A <sub>3</sub>	1.3901	1.3901	2.1356	7.8505

TABLE 21

FLUID PROPERTIES OF SUPERSONIC COMBUSTION RAMJET SHOCKED INLET, M =10,  $\mathcal{Y}_{0_2}^{\circ}$  =2

	DIFFUSER	COMBUSTION	CHAMBER	EXHAUST
	EXIT	INLET	EXIT	NOZZLE EXIT
	3c	<b>3</b> c	4	5
	322	302		
T (K)	2661	2661	2836.5	1839
P (atm)	.7	•7	7	.04
u (m/s)	1300.023	1300.023	1300.023	2407.716
M (frozen)	1.302	NC -	1.225	2.852
702	.186028	.190114	.1162	.14697
n <sub>N2</sub>	.769139	.714829	.713782	.747921
20	.015643	0	,025328	.000332
n N	.000001	0	NC PR	NC
7 NO	.029191	800 RES	.028587	.004120
<b>7</b> <sub>H2</sub>	1:0 · 3.54	.095057	.006309	.000099
1 <sub>H20</sub>	0	0	.071173	.098709
₹ OH	0	0	.029890	.001824
7 H	0	0	.008782	.000024
n (g/mole)	28.627296	NC	26.790837	27.59534
Y (frozen)	1.289737	NC	1.280166	1.286426
SR <sub>2</sub>	32.64504	NC	35.3137	35.31434
hf abs	3.700259	NC	3.79076	1.5194
AA3	1.0898	1.0898	1.2413	7.3824

TABLE 22

FLUID PROPERTIES OF SUPERSONIC COMBUSTION RAMJET
SHOCK EXPELLED (SUPERCRITICAL CONDITION)

	FREE STREAM	FUSELAGE AIRSTREAM 1	DIFFUSER INLET 2	DIFFUSER THROAT 3
T (K)	200	230	230	1107.5
P (atm)	.01	. 02	.02	6.28
u (m/s)	1420.3	1525	1525	657.56
M (frozen)	5	5	5	1.0103
702	.210085	.210084	.210084	.210033
7 <sub>N2</sub>	.789916	.789916	.789916	.789866
20	0	0	0	0
$n_{N}$	0	0	0	0
N <sub>NO</sub>	0	0	0	.000101
<b>7</b> <sub>H2</sub>	0	0	0	0
7 <sub>H20</sub>	0	0	0	0
₹ OH	0	0	0	0
₹ <sub>H</sub>	0 ·	0	0	0
$\mathcal{M}(g/\text{mole})$	28.853	28.853	28.853	28.85297
Y (frozen)	1.4	1.4	1.4	1.327386
SR <sub>2</sub>	NC	26.907	26.907	26.9061
hrabs R2T2	NC	3.496209	3.496209	17.77895
<b>Å</b> <sub>3</sub>	NA BERB	NA NA	28.1177	1

TABLE 23

FLUID PROPERTIES OF SUPERSONIC COMBUSTION RAMJET SHOCK EXPELLED, M =5,  $\mathcal{D}_{0_2}^{\circ}$ =0.5

	DIFFUSER	COMBUSTION C	CHAMBER	EXHAUST
	EXIT	INLET	EXIT	NOZZLE EXIT
2.13	3c	3c	4	5 (2)
T (K)	754.9	754.9	2604	1234.5
P (atm)	1.395	1.395	1.395	.02
u (m/s)	1110.205	1110.205	1110.205	2356.845
M (frozen)	2.064	NC	1.046	3.215
702	.210084	.147929	.008	.00001
<b>7</b> N <sub>2</sub>	.78916	.556213	.635607	.652770
20	0	0	.001783	0
2n	0	0	NC	NC
$n_{NO}$	0	0	.005034	.000002
$n_{\rm H_2}$	0	.295858	.027711	.000021
7H20	0	0	, 301 792	. 3471 96
2 OH	0	0	.014371	.000002
<b>≈</b> H	0	0	.005478	0
n(g/mole)	28.853	NC	23.98566	24.543573
Y (frozen)	1.329684	NC	1.247582	1.285460
<u>s</u> 2005	26.90625	NC	38.92074	38.92110
h <sub>f abs</sub>	11.74239	NC	15.928338	-25.9767
A A	1.8175	1.8175	7.5414	114.7979
		56		

TABLE 24

FLUID PROPERTIES OF SUPERSONIC COMBUSTION RAMJET SHOCK EXPELLED, M =5,  $\mathcal{V}_{0_2}^{\circ}$ =2

	DIFFUSER	COMBUSTION	CHAMBER	EXHAUST
	EXIT	INLET	EXIT	NOZZLE EXIT
	3c	3c	4	5
T (K)	936	936	1795	484
P (atm)	3.2	3.2	3.2	.02
u (m/s)	910.123	910.123	910.123	1742.997
M (frozen)	1.515	NC	1.119	3.889
702	.210084	.190114	.14825	.1497
7 <sub>N2</sub>	.789916	.714829	.749144	.750499
70	0	0	.000010	0
₹ <sub>N</sub>	0	0	NC	NC
₹ <sub>NO</sub>	0	0	.002600	0
<b>n</b> <sub>H2</sub>	0	.095057	.000003	0
1 H20	0	0	.099657	.099800
<b>≉</b> <sub>OH</sub>	0	0	.000266	0
<b>₹</b> H	0	0	.0000002	0
n(g/mole)	28.853	NC	27.610155	27.613586
Y (frozen)	1.33755	NC	1.289948	1.378425
<u>s</u> R <sub>2</sub>	26.9074	NC	30.9574	30.9551
h <sub>f abs</sub>	14.79195	NC .	17.73222	-5.186886
A 3	1.1983	1.1983	2.2811	54.0926

TABLE 25

FLUID PROPERTIES OF SUPERSONIC COMBUSTION RAMJET SHOCK EXPELLED (SUPERCRITICAL CONDITION)

	FREE	FUSELAGE	DIFFUSER	DIFFUSER
	STREAM	AIRSTREAM	INLET	THROAT
	00	1	2	3
T (K)	200	300	300	2940
P(atm)	.01	. 04	. 04	272
u (m/s)	2840.5	2785	2785	1042.96
M (frozen)	10	8	8	0.999
<b>7</b> 02	.210084	.210084	.210084	.187123
<b>7</b> <sub>N2</sub>	.789916	.789916	.789916	.767460
70	0	0	0	.002396
n	0	0	0	.0000005
<b>Z</b> NO	0	0	0	.043021
<b>₹</b> <sub>H2</sub>	0	0	0	0
<b>7</b> <sub>H2</sub> 0	0	0	0	0
$n_{\text{OH}}$	0	0	0	0
₹ H	0	0	0	0
m(g/mole)	28.853	28.853	28.853	28.818412
8 (frozen)	1.4	1.4	1.4	1.284703
<u>S</u> R <sub>2</sub>	NC	27.5823	27.5823	27.5793
ф				7.41 <u>1841</u>
hf abs	NC	3.49805	3.49805	42.0671
R <sub>2</sub> T <sub>2</sub>	NA.	NA CBO	259.3541	1
A <sub>A</sub>		5.5	~57.55.1	

TABLE 26

FLUID PROPERTIES OF SUPERSONIC COMBUSTION RAMJET SHOCK EXPELLED, M =10,  $\mathcal{J}_{0_2}^{\circ}$ =0.5

	DIFFUSER	COMBUSTION	CHAMBER	EXHAUST
	EXIT	INLET	EXIT	NOZZLE EXIT
	3c	3c	4	5
T (K)	2570	2570	3579.4	729
P (atm)	136	136	136	. 04
u (m/s)	1480.836	1480.836	1480.836	3423.661
M (frozen)	1.516	NC	1.176	5.969
202	.19657	- 147929	.099	.0000001
<b>7</b> <sub>N2</sub>	.776561	.556213	.613449	.652778
<b>7</b> 0	.000765	0	.005079	0
₹ <sub>N</sub>	0	0	NC	NC
<b>≈</b> <sub>NO</sub>	.026104	0	.017033	0
<b>1</b> <sub>H2</sub>	0	.295858	.051814	0
<b>n</b> <sub>H2</sub> 0	0	.255195	.347222	.347222
<b>₹</b> OH	0	0	.033965	0
<b>N</b> H	0	0	.013684	0
n(g/mole)	28.841937	NC	23.389112	24.543781
Y (frozen)	1.288146	NC NC	1.245256	1.332298
<u>s</u> R <sub>2</sub>	27.58286	NC	33.8596	33.85376
hf abs	35.67526	NC	39.433868	-28.35987
A <sub>3</sub>	1.2303	1.2303	2.1131	603.1073

FLUID PROPERTIES OF SUPERSONIC COMBUSTION RAMJET SHOCK EXPELLED, M =10,  $\mathbf{v}_{0_2}^{\circ}$ =2

	DIFFUSER	COMBUSTION C	HAMBER	EXHAUST
	EXIT	INLET	EXIT	NOZZLE EXIT
	3c	3c	4	5
T (K)	2734	2734	3110	448 (mts)
P (atm)	186	186	186	. 04 (a)(m)
u (m/s)	1309.282	1309.282	1309.282	2752.387
M (frozen)	1.300	NC Establish	1.197	6.368
702	.19268	.190114	.1128	.1497
n <sub>N2</sub>	.772792	.714829	.724572	.750499
<b>7</b> 0	.001331	0	.004156	0
₹ <sub>N</sub>	.0000001	0	NC	NC ON TO
$nall_{NO}$	.033197	0 658562	.041401	0
<b>₹</b> H <sub>2</sub>	0	.095057	.001223	0
<b>7</b> <sub>H<sub>2</sub>0</sub>	0	0	.089893	.099800
<b>₹</b> OH	00 4866.50	0	.015420	0 5
7 H	0 94108640	0	.000559	0 (minu/21)2/4
n(g/mole)	28.83377	NC	27.42287	27.614384
Y (frozen)	1.286661	NC	1.269813	1.385055
<u>s</u> R <sub>2</sub>	27.58312	NC	29.94482	29.94014
h <sub>f abs</sub>	38.4437	NC	39.38418	-4.431565
<u>A</u> 3	1.0827	1.0827	1.2950	409.7627

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